



# Baseline Testing of the Hybrid Electric Transit Bus

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# BASELINE TESTING OF THE HYBRID ELECTRIC TRANSIT BUS

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## SUMMARY

A government, industry and academic cooperative is developing an advanced hybrid electric city transit bus. Goals of this effort include doubling the fuel economy of buses currently in service and reducing emissions to one tenth of EPA standards. Unique aspects of the vehicle's power system include the use of ultra-capacitors for the energy storage system, and the planned use of a natural gas-fueled turbogenerator, developed from a small jet engine. At over 17000 kg gross weight, this is the largest vehicle to ever use ultra-capacitor energy storage. Power from both the auxiliary power unit and the energy storage system is delivered to a variable speed electric motor that drives the rear axle.

The bus uses regenerative braking to improve fuel efficiency. This technology recovers much of the kinetic energy of the vehicle during deceleration. This replenishes the energy storage system each braking cycle and extends the life of the mechanical brakes.

The bus was tested at the Transportation Research Center in East Liberty, Ohio between September 21-25, 1998. Tests were performed to characterize the vehicle's performance using industry standard drive cycles. This paper describes the HETB vehicle, the results of performance tests, and future plans for vehicle development.

## INTRODUCTION

The NASA Lewis Research Center initiated the Hybrid Electric Transit Bus project as an excellent opportunity to transfer technology from the aerospace and military industries to a commercial venture. The project is seen as a way to reduce pollution in urban areas, reduce fossil fuel consumption and reduce operating costs for mass transit systems.

The NASA Lewis Research Center provides overall project coordination, expertise in system modeling and experience selecting key components. NASA Lewis is responsible for developing the vehicle power control system. This includes charge and discharge circuitry for the energy storage system and control algorithms. Wherever practical, off-the-shelf components have been integrated into the power control system.

The Greater Cleveland Regional Transit Authority (RTA) is a large urban transit authority that operates over 700 buses in greater Cleveland. RTA is a good example of a potential customer. It is located near the other partners and has aggressively pursued new technologies to reduce emissions. RTA has developed the infrastructure to support alternative fuels, especially natural gas. RTA supplied the project with a vehicle and has supported the subsequent modifications and testing.

Bowling Green State University (BGSU), College of Technology, is a leader in the development of electric vehicle drive trains. BGSU and the Lincoln Electric, Inc., Motors Division, developed the traction and auxiliary motors. BGSU engineered and assembled the major drive train components onto the rear engine cradle of the bus.

Howard University (Washington, DC) is responsible for developing advanced energy management algorithms using neural network technology. This may eventually offer more fuel-efficient operation of hybrid vehicles by adjusting the state of the energy system based on the recognition of variable route conditions. The project team plans to incorporate these algorithms into the vehicle controller at a later date.

Flxible, Inc., has historically been the dominant producer of forty-foot transit buses. This duty class of urban transit buses is the mainstay of the industry. Flxible became a market leader by producing an advanced semi-monocoque aluminum-bodied vehicle at the lowest cost. Although the forty-foot Flxible bus is currently out of production, Flxible supported the vehicle engineering and integration. Since Flxible is no longer producing buses, the project is seeking other vehicle manufacturers to embrace the technology and either manufacture hybrids or retrofit conventional vehicles to a hybrid configuration.

## TEST OBJECTIVES

The Department of Transportation's Transit Bus "White Book" was used as a basis for the hybrid electric transit bus testing that was performed at the Transportation Research Center. Of particular interest are the following characteristics: fuel economy, vehicle speed, acceleration time, average acceleration, maximum jerk, gradeability, sound level, range over stop-and-go driving schedules, and regenerative braking results. The performance of the various vehicle components, especially the motors, controllers, auxiliary power unit (APU), and energy storage system are also of interest.

## TEST VEHICLE DESCRIPTION

The Hybrid Electric Transit Bus (HETB) is a converted forty-foot Flxible transit bus. The vehicle is shown in Figs. 1 and 2 and described in detail in Appendix A. The HETB is a series-hybrid as shown in Fig. 3. Series-hybrids convert all power produced by the generator to electric power. The combination of electric power from the engine-generator and the energy storage system power the single centrally located variable-speed electric motor that is attached to the rear differential/axle. The engine-generator and energy storage systems power all of the auxiliary systems of the bus as well. These include lighting, heating, pneumatics, hydraulics, and other vehicle systems. The engine-generator combination is also referred to as the auxiliary power unit (APU).

The energy storage system uses a 2100 lb. bank of thirty ultra-capacitors to store electrical energy. The capacitor bank is shown in Fig. 4. The capacitor bank is capable of storing 1.6 MJ of energy (20 Farads at 400 Volts). This state-of-the-art technology not only has much longer life than conventional batteries, but also provides exceptional capability to recover energy that would otherwise be lost during braking. The HETB is the largest vehicle ever to use ultra-capacitor energy storage.

Fig. 1 – Hybrid Electric Transit Bus

NASA  
C-98-1482



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Fig. 2 – HETB Rear Cradle

NASA  
C-98-1481



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Fig. 3 – HETB Schematic Diagram

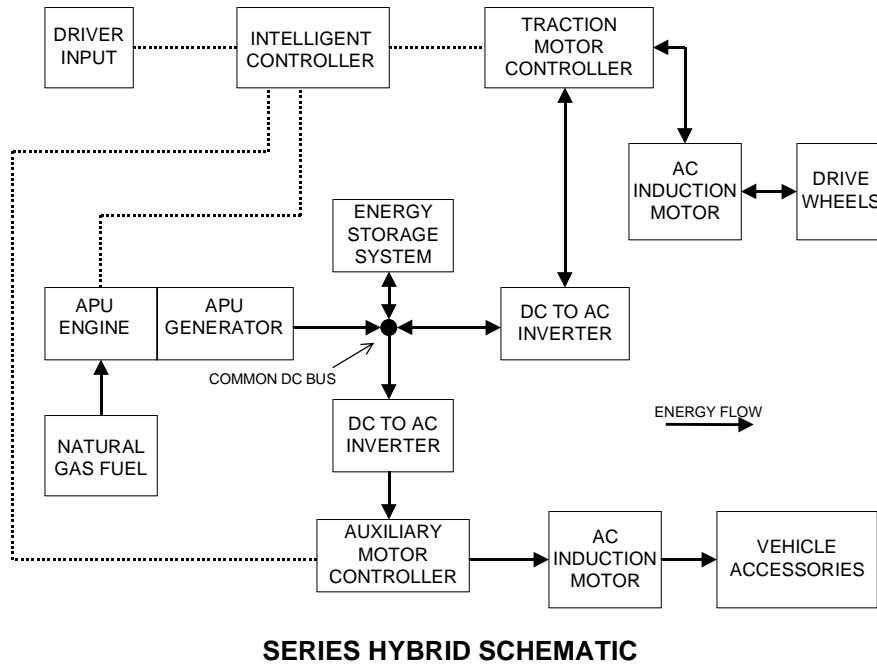


Fig. 4 – Ultra-Capacitor

NASA  
C-97-1248



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The electric traction motor is a four-quadrant, vector-controlled AC induction motor. Induction motors are very reliable. Vector motor controllers allow for independent and efficient torque control over a wide speed range. This Lincoln Electric motor is designed with a light alloy frame. BGSU-designed the motor's oil cooling system. The motor produces 200 HP but weighs only 350 lb. An auxiliary motor drives the other subsystem loads, such as air and hydraulic pumps.

The power management and data acquisition system consists of a programmable logic controller (PLC), a laptop computer and a communication bus. The PLC sequences the various subsystems at startup and shutdown. The laptop computer controls the operation of the APU in response to the state of discharge of the energy storage system. The laptop PC also acts as an operator interface. Optimizing the state of the energy storage system during the driving cycle allows for full recovery of energy during braking. This enhances vehicle performance. The response to the driver's acceleration and brake controls is similar to a conventional vehicle.

For these tests, the vehicle's APU is rated at 50 kW over a voltage range of 250 V to 360 V. The engine is a 5-cylinder inline, 2.3-liter displacement. It was modified to run on compressed natural gas (CNG). The generator is a 12-pole, wound field machine with 3-phase rectified output. It delivers rated voltage over a speed range of 2100 to 6000 rpm.

The APU controller receives a power setpoint command from the laptop computer. It then determines the optimum engine speed for that power level and moves the engine to that speed. The speed/load curve is programmed into the APU controller to produce power at the point of lowest emissions and greatest fuel economy. The APU controller will shut the APU down under certain fault conditions, such as low oil pressure, high coolant temperature, engine over-speed and high/low voltage.

The choice of the energy storage system and its electrical characteristics strongly influences the control strategy used. Ultra capacitors have several properties that are significantly different from the chemical batteries typically used on electric vehicles.

First, the state of charge of the capacitors may be very accurately determined from the measured terminal voltage. This is a significant advantage over batteries, whose relationship between state of charge and terminal voltage is highly non-linear. Batteries also exhibit hysteresis in their voltage, current and state of charge relationships. Batteries, therefore, require a much more sophisticated control system to manage state of charge.

Another difference between batteries and capacitors is that batteries must be current limited and/or cell voltage limited. This is especially true during the charging cycle, and it becomes critical as the battery approaches a full charge. Near full charge, lead acid and many other chemical batteries cannot accept high currents without plate damage. To prevent loss of battery life, additional controls are required. This further complicates the control system. On the other hand, capacitors can accept very high currents. They approach their voltage limit more slowly and do not experience damage while accepting currents just below full charge.

Batteries do have a much greater energy storage capacity than capacitors. Although the capacitor bank on the HETB is quite large, much of its energy is consumed during a single sustained acceleration.

In order to compare the performance of these two energy storage options, the HETB was temporarily equipped with two series strings of twenty-eight 12 V batteries. These batteries are Optima D750S, 12 V, 50 A-hr deep cycle units. These are sealed, maintenance-free, absorbed glass mat batteries, representing the state-of-the-art in lead acid battery technology.

When the capacitors are used for energy storage, the laptop computer executes a proportional-integral (PI) control algorithm to maintain the capacitor voltage as close as possible to a predetermined setpoint (325-350 V). This is accomplished by varying the power requested from the APU. The control algorithm is described in detail in Appendix F. When batteries are used, an operator manually controls the power requested to the APU from the laptop in order to keep the battery voltage near a setpoint of 336 V.

## INSTRUMENTATION

The HETB was instrumented to measure vehicle speed, distance, acceleration, jerk and sound pressure levels. Additional channels measured the APU voltage, as well as the following currents: traction motor, auxiliary motor, energy storage system and generator. Temperatures were measured at the APU, the ultra-capacitors, the rear cradle, traction motor stator, traction motor controller, auxiliary motor stator and the outside ambient temperature. Most of these data were sent to an on-board digital data acquisition system, sampled at 100 Hz and recorded on digital tape. The APU voltage was sampled at 1 Hz and stored on the laptop PC, which was externally synchronized with the data acquisition system. The instrumentation configuration is described in Appendix B.

Power for the data acquisition system, was derived from the vehicle's 12 V starting, lighting and ignition (SLI) battery.

The fuel economy tests were conducted using a small auxiliary tank that was filled with CNG prior to the test. The tank was initially weighed and then installed. Following the test, the tank was weighed again to determine fuel consumption.

## TEST PROCEDURES

The tests described in this report were conducted at the Transportation Research Center in East Liberty, Ohio. A description of the track is given in Appendix C. The tests were conducted in accordance with the White Book Technical Specifications for Wheelchair-Accessible 40-Foot Transit Coaches, provided in Appendix D.

## TEST RESULTS

### Vehicle Performance

Eleven tests were conducted to determine vehicle performance, per Table 1:

**Table 1 – Performance Tests Conducted on the Hybrid Electric Transit Bus**

<i>Test Number</i>	<i>Energy Storage System Used</i>	<i>Regenerative Braking Used?</i>	<i>Top Vehicle Speed</i>	<i>Driving Cycle</i>
1	Capacitors	No	15 mph	7 CBD cycles, Fuel Economy
2	Capacitors	Yes	15 mph	7 CBD cycles, Fuel Economy
3	Capacitors	No	15 mph	2 CBD cycles
4	Capacitors	Yes	15 mph	2 CBD cycles
5	Batteries	No	15 mph	2 CBD cycles
6	Batteries	Yes	15 mph	2 CBD cycles
7	Batteries	No	20 mph	2 CBD cycles
8	Batteries	Yes	20 mph	2 CBD cycles
9	Batteries	No	35 mph	2 Arterial cycles
10	Batteries	Yes	35 mph	2 Arterial cycles
11	Batteries	No	35 mph	Acceleration test to 35 mph

A similar set of plots have been included in Appendix E for each of the tests:

- Vehicle Speed, system voltage, and component currents vs. elapsed time
- Vehicle Speed, system voltage, and component powers vs. elapsed time
- Component temperatures vs. elapsed time
- Vehicle Speed and acceleration vs. elapsed time
- Vehicle Speed, APU power requested, APU power delivered and APU engine speed vs. elapsed time

A summary of the test results is shown in Table 2 at the end of this section.

During the test period the winds were light. There were no indications that the winds affected the test results.

### Maximum Speed

The maximum speed of the vehicle was measured to be 40 mph in first gear during the preliminary checkout. The maximum speed is defined as the average speed that could be maintained on the track under full power. No maximum speed measurement was made in second gear.

## Fuel Economy

The fuel economy of the vehicle was determined from the following information:

*Air density = 0.0807 lb/scf*

*Air density relative to fuel = 0.578*

*Diesel fuel equivalent: 100 scf of CNG = 0.748 gal diesel*

$$MPG_{diesel} = \frac{\text{miles}}{\text{lb}_{CNG\_used}} \times \frac{0.0807 \text{ lb}}{\text{scf}_{air}} \times \frac{0.578 \text{ scf}_{air}}{\text{scf}_{CNG}} \times \frac{100 \text{ scf}_{CNG}}{0.748 \text{ gal}_{diesel}}$$

Capacitors were used as the energy storage system for the fuel economy tests. Two fuel economy tests were conducted, one with regenerative braking and one without. The CBD driving cycle was used over a four-mile route (28 cycles).

The results of the two fuel economy tests were 3.01 mpg with no regenerative braking and 3.65 mpg with regenerative braking. This translates to an improvement of 21.2% with the use of regenerative braking. These results are compared with a typical diesel vehicle in Fig. 5.

## Acceleration

The average acceleration,  $a_n$ , of the vehicle is computed as a change in vehicle speed as a function of time.

$$a_n = \frac{V_n - V_{n-1}}{t_n - t_{n-1}}$$

Acceleration times are given in Table 2. As shown in Fig. 6, the HETB traction control system is capable of meeting the White Book acceleration specification up to 35 mph in first gear. This particular test was conducted using the batteries.

## Gradeability

The maximum specific grade, in percent, that a vehicle can climb at a particular speed,  $v$ , was determined from maximum acceleration tests using the equation:

$$G = 100 \tan (\sin^{-1} 0.0455 a_n)$$

for  $v$  mph.

Fig. 5

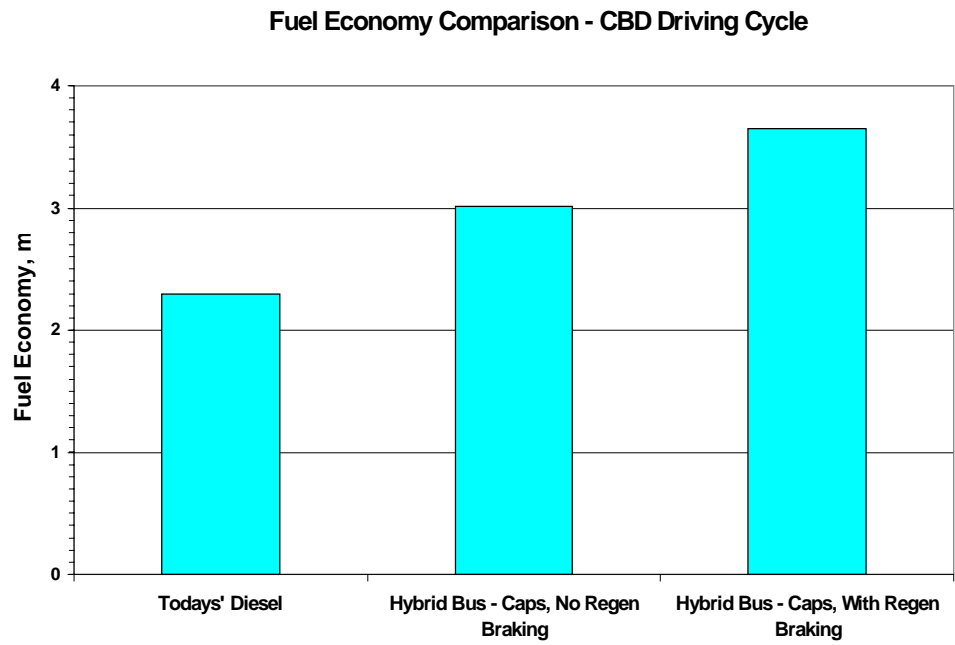
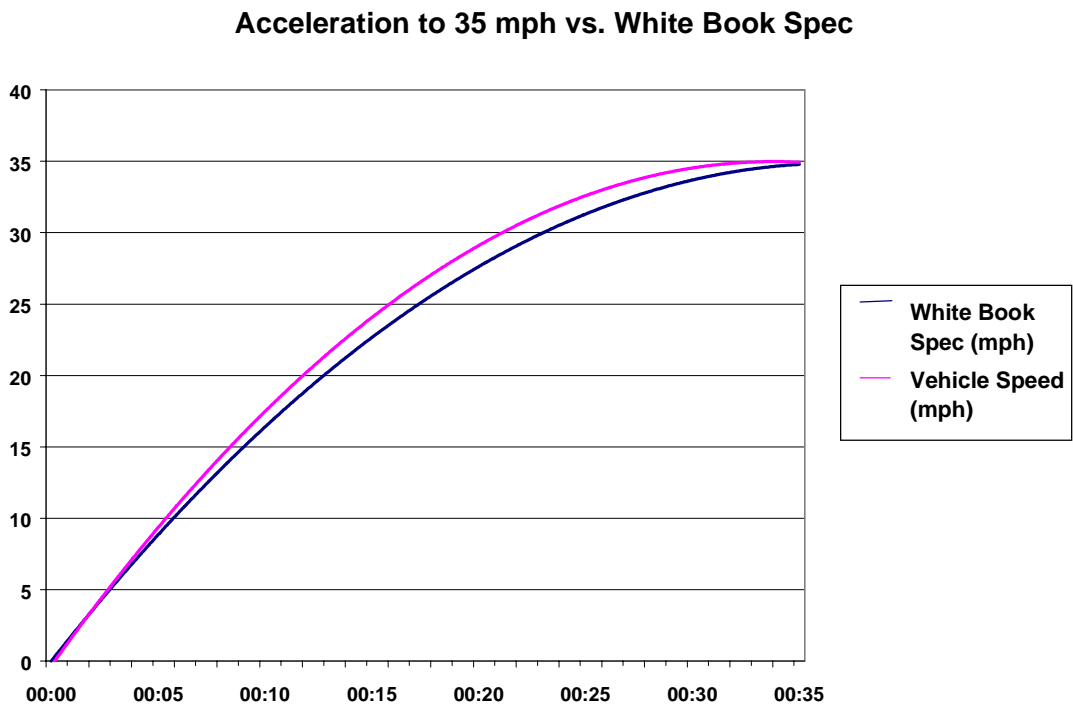


Fig. 6



## **Sound Pressure Level**

The sound pressure level of the vehicle was monitored inside the vehicle. The sensor was positioned behind the driver at ear level, or 45 inches above the floor. For a one-mile (7-cycle) CBD test with capacitor energy storage and regenerative braking, the maximum sound pressure level was 73 dBA and the average sound level was 40 dBA., as shown in Fig. 7. The sound level is highest during and just after acceleration, when the APU and drive train are providing maximum power output. The sound level is much lower when the vehicle is idle.

## **Range**

The range of the vehicle was determined from the fuel economy data. The vehicle has three CNG fuel tanks. Each tank holds 2683 scf of CNG at 3600 psi. The total fuel weight for a fully-fueled vehicle is 375.5 lb.

Without regenerative braking, the fuel economy is 2.0675 lb/min. This yields a range of 181.6 miles. With regenerative braking, the fuel economy is 1.705 lb/min. This yields a range of 220.0 miles.

## **Braking**

Identical tests were conducted, both with and without regenerative braking, to determine its effectiveness. It appears that regenerative braking is more effective with capacitors. The benefits of regenerative braking with batteries are highly dependent on the state of charge of the batteries.

From an initial speed of 15.4 mph, the regenerative braking distance with capacitors was 103 feet, and the braking time was 10 seconds. The regenerative braking distance with batteries was 157 feet, and the braking time was 14 seconds. These results are plotted in Fig. 8.

Regenerative braking alone is sufficient to stop the vehicle in a Central Business District test using capacitors. A combination of regenerative and mechanical braking is required for batteries.

## **Summary**

An overall summary of the vehicle testing is shown in Table 2.

Fig. 7

Caps, CBD Fuel Economy, 15 mph, Regen Braking  
Sound Pressure Level

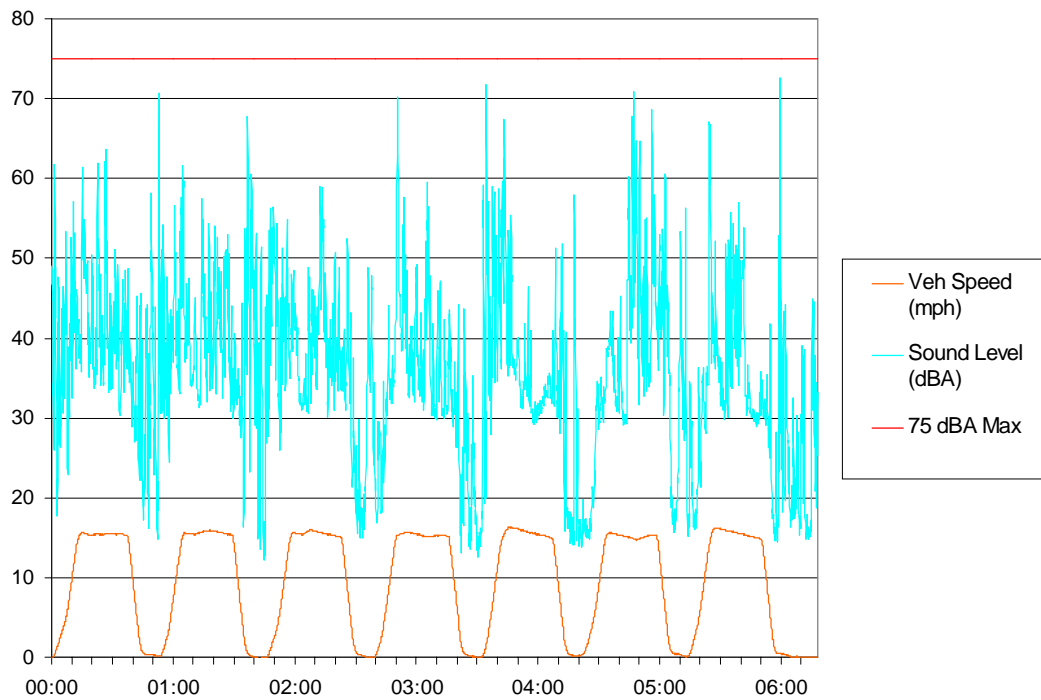
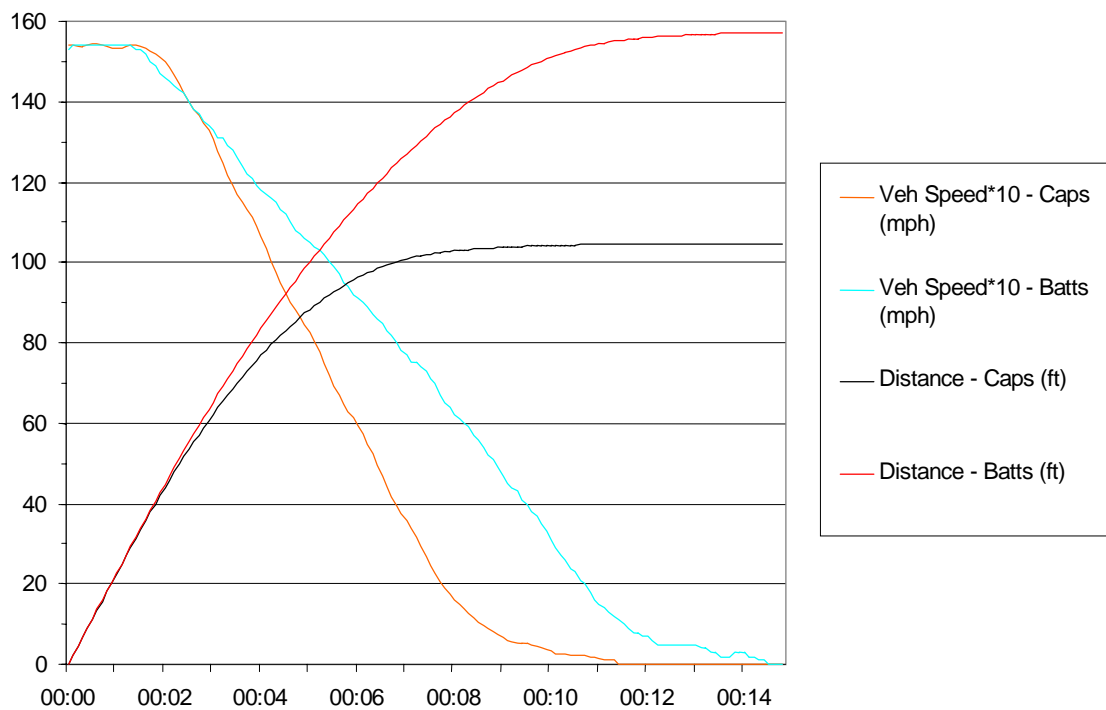


Fig. 8

Regen Braking Performance, Caps vs. Batteries  
Stop From 15.4 mph



**Table 2 – Summary of Test Results for the Hybrid Electric Transit Bus**

Parameter	Configuration	Test Results	Remarks
Fuel Economy	Capacitors, No Regen Br ----- Capacitors, With Regen Br	3.01 mpg diesel ----- 3.65 mpg diesel	4 miles CBD, (28 cycles) @ 15 mph
Top Speed	Batteries	40 mph	Limited by maximum motor speed in first gear
Acceleration Times			
10 mph	Capacitors ----- Batteries	8 sec ----- 6 sec	
15 mph	Capacitors ----- Batteries	11 sec ----- 9 sec	
20 mph	Batteries	12 sec	
30 mph	Batteries	22 sec	
35 mph	Batteries	35 sec	
Average Acceleration	Capacitors ----- Batteries	0.062 g ----- 0.076 g	0-15 mph
Maximum Acceleration Jerk	Capacitors ----- Batteries	0.04 g/s ----- 0.06 g/s	0-15 mph
Maximum Deceleration Jerk	Capacitors ----- Batteries	-0.06 g/s ----- -0.03 g/s	With Regenerative Braking
Gradeability	Capacitors ----- Batteries	2.77 % ----- 3.39 %	At 15 mph
Sound Level	Capacitors	73 dBA max ----- 40 dBA avg	1 mile CBD, (7 cycles) @ 15 mph
Range	Capacitors, No Regen Br ----- Capacitors, With Regen Br	181.6 miles ----- 220.2 miles	2683scf CNG @ 3600 psi
Regenerative Braking Distance	Capacitors ----- Batteries	103 ft ----- 157 ft	From 15.4 mph No mechanical braking used
Regenerative Braking Time	Capacitors ----- Batteries	10 sec ----- 14 sec	From 15.4 mph No mechanical braking used



## CONCLUDING REMARKS

The HETB as tested and described in this report is a proof-of-concept prototype with several known limitations. A conventional bus, fully loaded with passengers, has a weight of around 33,000 lb. The HETB had a measured weight of 37,600 lb. This extra weight, mostly attributed to having redundant energy storage systems, had a significant impact on vehicle performance. The size and weight of commercially available capacitors limited the amount of capacitance that could be installed. The APU was designed for use with lead acid batteries, not optimized for capacitors. Also its 39 kW maximum output power was known to be insufficient.

It was difficult to accelerate the vehicle to above 20 mph on capacitors. However, the data suggest that raising the total capacitance and the APU output power is all that is required to match the performance of the battery energy storage system. Under battery power, the drive train performed very well in terms of acceleration and top speed in first gear.

The test results did clearly demonstrate that capacitors are superior to batteries in delivering load currents to the traction motor when accelerating. They are also better at accepting regenerative braking currents, allowing for less usage of the mechanical brakes. The project team believes that the advantages of capacitors warrant further development efforts.

Future plans for the HETB call for the replacement of the present APU with a turbine driven APU, increasing the capacitor energy storage bank while reducing its present weight and volume, redesigning the gearbox with integrated hydraulic circuits and developing a fully integrated power management and control system.

## REFERENCES

1. "Baseline Advanced Design Transit Coach Specifications", Department of Transportation, 1978.
2. Viterna, L.A., "Hybrid Electric Transit Bus", NASA Technical Memorandum 113176, October, 1997.

## APPENDIX A

### VEHICLE SUMMARY DATA SHEET

1.0 Vehicle Manufacturer	Flxible, Inc., Delaware, OH
2.0 Vehicle	Flxible Metro Model 40102 Conversion
3.0 Vehicle Configuration	Series Hybrid with Regenerative Braking
4.0 Traction Motor	
4.1 Traction Motor Configuration	3-phase Induction
4.2 Traction Motor Horsepower	200 HP
4.3 Traction Motor Speed Range	0-10000 rpm
4.4 Traction Motor Cooling	Liquid cooled
5.0 Drivetrain	
5.1 Transmission Type	2 speed manual
5.2 Gear Ratios	7.6 to 1 First Gear, 4.8 to 1 Second Gear
5.3 Axle Ratio	4.7 to 1
5.4 Final Drive Ratio	35.5 to 1 First Gear, 22.4 to 1 Second Gear
6.0 Auxiliary Power Unit (APU)	
6.1 Engine Configuration	5-cylinder In-line
6.2 Engine Displacement	2.3 liter
6.3 Engine Horsepower	90 HP
6.4 Engine Speed Range	2100 to 4500 rpm
6.5 Engine Fuel	Compressed Natural Gas (CNG)
6.6 Generator Type	12 Pole, 3-phase, Wound Field
6.7 Generator Power	50 kW
6.8 Generator Voltage	250 to 360 VDC
7.0 Vehicle Dimensions	
7.1 Wheel Base	299 in (7.59 m)
7.2 Length	40 ft, 8.4 in (12.41 m)
7.3 Track	85.66 in (2.18 m) front, 76.50 in (1.94) rear
7.4 Width	102 in (2.59 m)
7.5 Height	121.25 in (3.08 m)
7.6 Base Curb Weight	26054 lb (11814 kg)
7.7 Total Weight (as tested)	37600 lb (17055 kg)
7.8 Fuel Capacity	8050 scf total @ 3600 psig max

## 8.0 Accessories

8.1 Auxiliary Motor Configuration	3-phase Induction
8.2 Auxiliary Motor Horsepower	25 HP
8.3 APU Fan Motor Configuration	3-phase Induction
8.4 APH Fan Motor Horsepower	3 HP
8.5 Accessory and APU Motor Cooling	Totally Enclosed Air Cooling

## 9.0 Energy Storage

### 9.1 Capacitors

9.1.1 Configuration	Bank of 30 ultra-capacitors (15 legs of two capacitors in series)
9.1.2 Capacitance	2.5 F each, 18.75 F total
9.1.3 Energy Rating	50 kJ each, 1.6 MJ total
9.1.4 Voltage Rating	200 V each
9.1.5 Dimensions	9 in (229 mm) diameter x 12 in (305 mm) length
9.1.6 Weight	70.5 lb (32 kg) each, 2115 lb (959 kg) total

### 9.2 Batteries

9.2.1 Configuration	Two banks of 28 batteries each
9.2.2 Energy Rating	180 kJ each, 5.04 MJ per bank
9.2.3 Voltage Rating	12 V each, 336 V per bank
9.2.4 Dimensions	10.0 in (254 mm) x 6.8 in (173 mm) x 7.9 in (200 mm)
9.2.5 Weight	45 lb (20.4 kg) each, 2520 lb (1143 kg) total (both banks)

## APPENDIX B

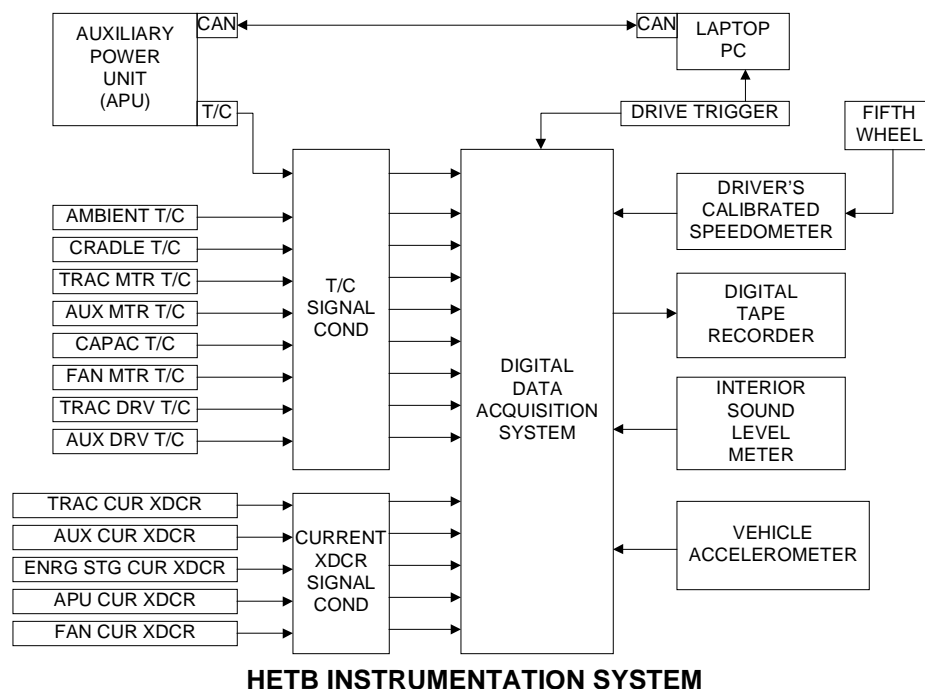
### DESCRIPTION OF THE INSTRUMENTATION SYSTEM

A block diagram of the HETB instrumentation system is shown in Fig. B-1.

The APU has an integral instrumentation system that monitors APU output voltage, output current, engine speed, coolant temperature and oil pressure. These data are sampled at 1 Hz and transmitted to the laptop PC via a controller area network (CAN) interface. The PC logs the APU data. A type J thermocouple was mounted on the APU to monitor its temperature.

All other measurements were obtained with a MegaDAC data acquisition system, sampling at 100 Hz. Type J thermocouples were used for all temperature measurements. Hall effect transducers were used for all current measurements. A fifth wheel installed on the back of the HETB provided vehicle speed and distance measurements. These data were sent to a calibrated speedometer for the driver, as well as to the data acquisition system. An accelerometer mounted near the vehicle's center of mass provided the acceleration and jerk data. A sound pressure level sensor was positioned behind the driver's seat at ear level, 45 inches from the floor. The laptop PC was externally synchronized with the data acquisition system.

Fig. B-1



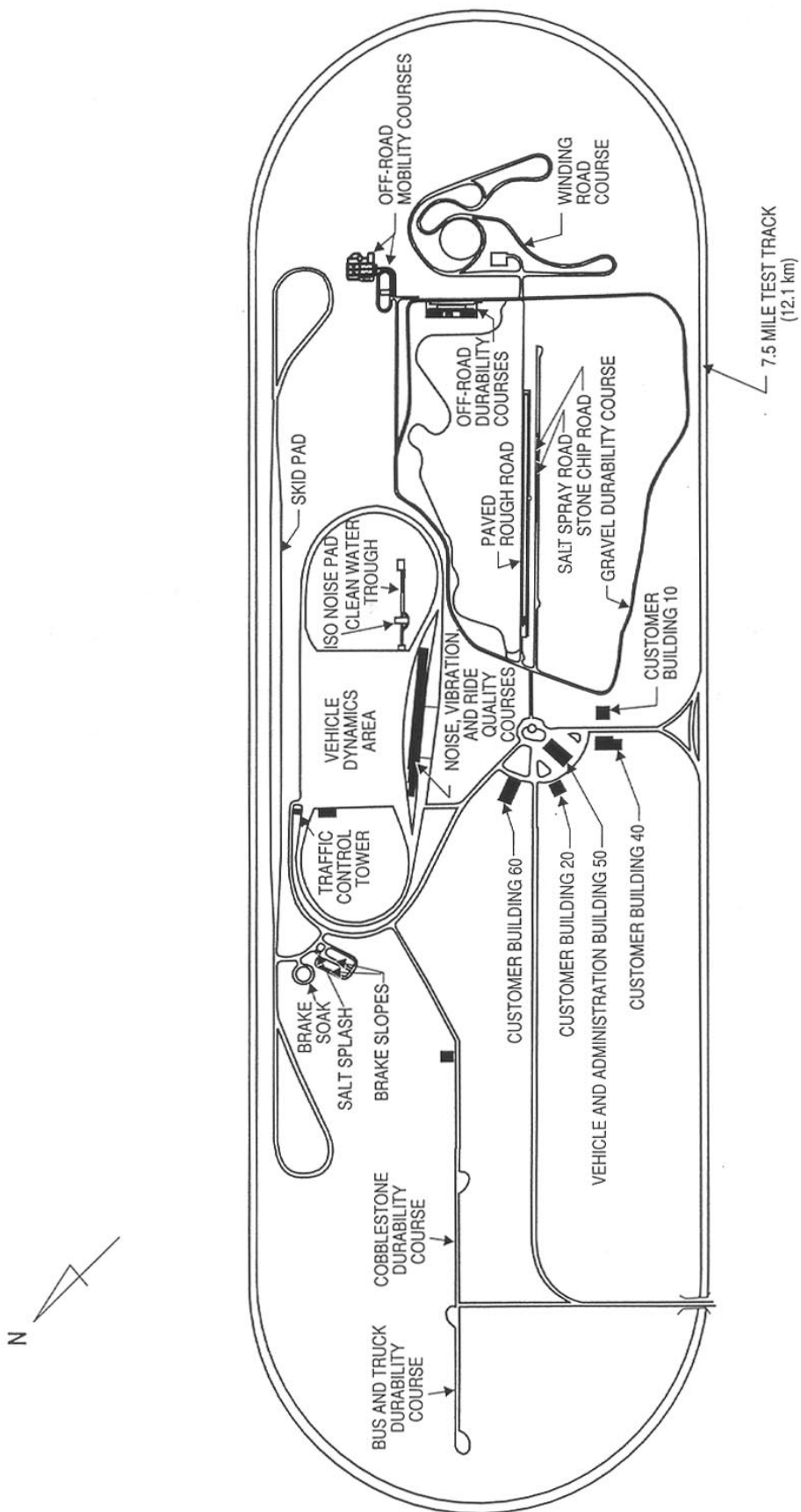
## APPENDIX C

### DESCRIPTION OF VEHICLE TEST TRACK

The track used to conduct the tests described in this report is the Transportation Research Center, located in East Liberty, Ohio. A facility map is shown in Fig. C-1.

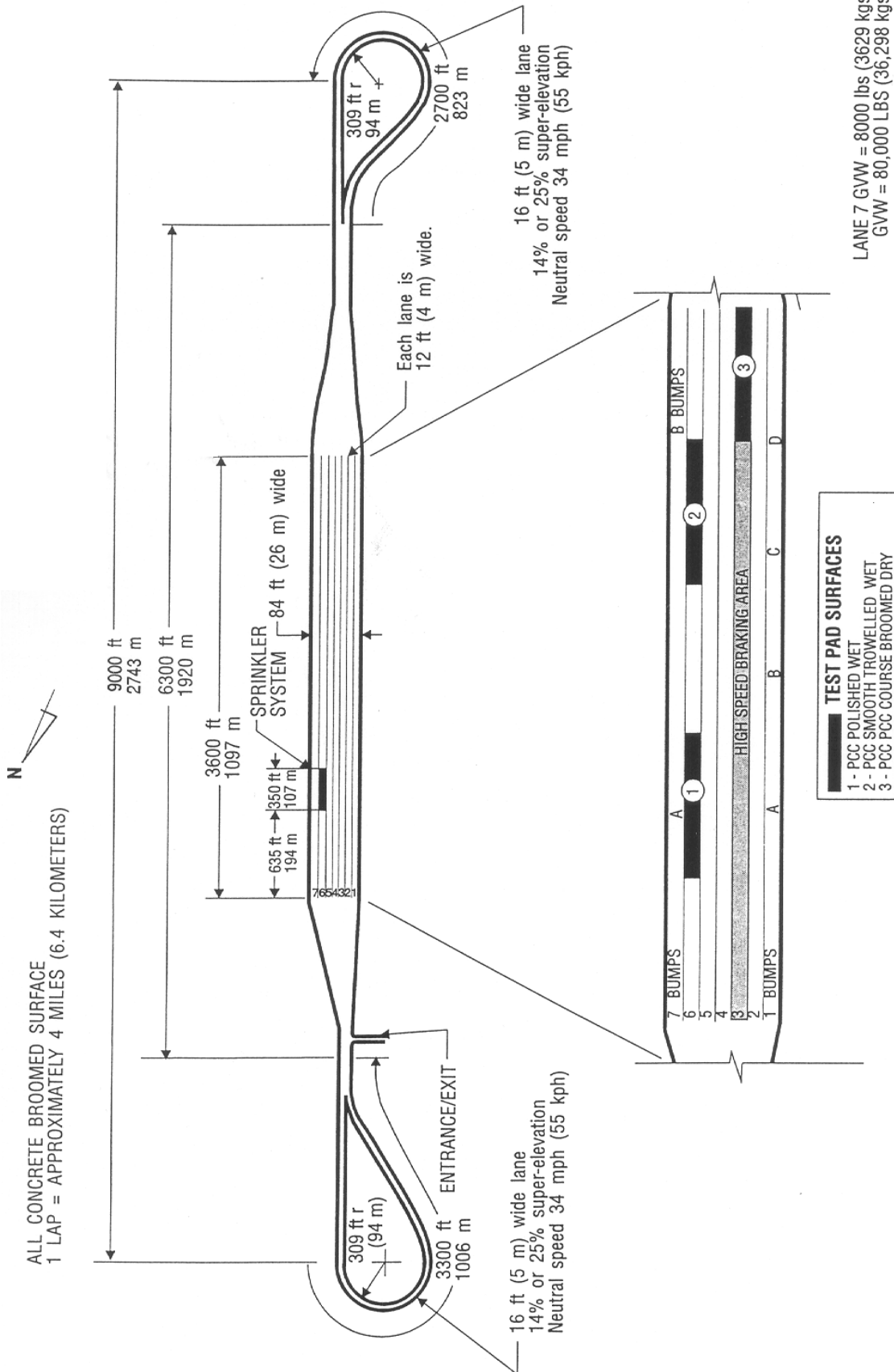
Preliminary tests were conducted on the skid pad. This concrete broomed track is a four-mile loop. The skid pad is shown in Fig. C-2.

Tests documented in this report were conducted on Lane 1 of the 7.5-mile test track. This lane is concrete with ten degree banking on the turns. This track is shown in Fig. C-3.



NOT TO SCALE

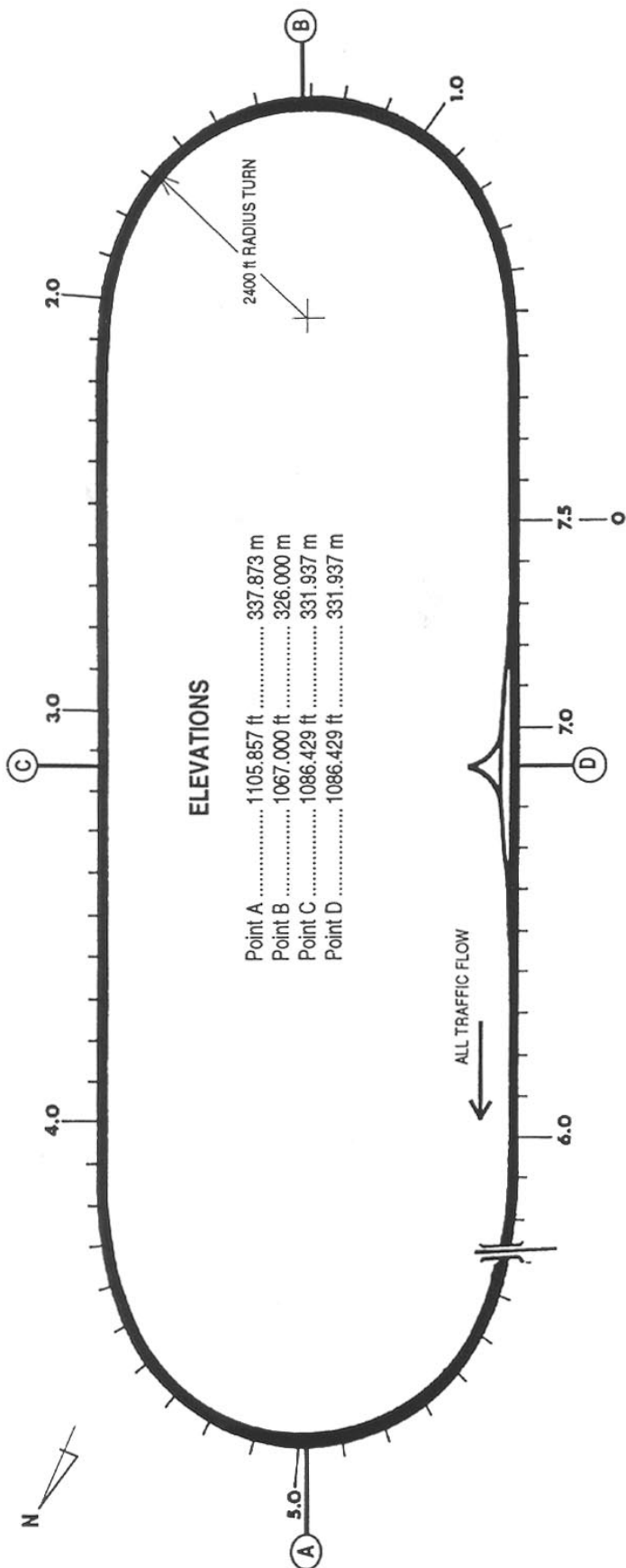
## TEST FACILITY DETAIL



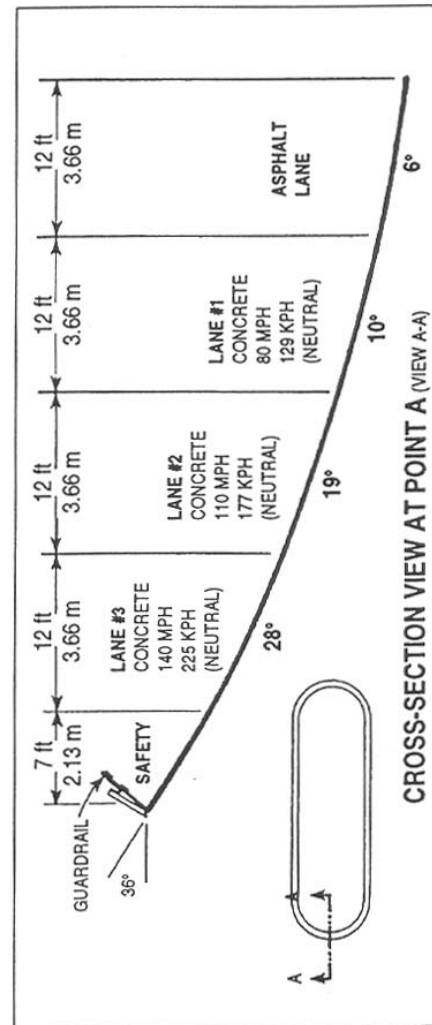
NOTE: BUMP COURSES PARALLEL THE PERIMETERS OF LANES 1 AND 7.

## SKID PAD

Not to scale  
All dimensions are approximate



GWV = 80,000 LBS (36,298 kgs)



### DISTANCES

Lane 3	7.539 mi	12.133 km
Lane 2	7.521 mi	12.104 km
Lane 1	7.507 mi	12.081 km
Point A to Point B	3.333 mi	5.364 km
Point C to Point D	.947 mi	1.524 km

NOT TO SCALE

### 7.5-MILE TEST TRACK



## APPENDIX D

### DESCRIPTION OF TEST CYCLES

Testing of the HETB was based on the Department of Transportation Transit Bus White Book Specification (reference 1). The test matrix shown in table D-1 was derived from the White Book. The transit bus operating duty cycle is shown in Table D-2. The Central Business District (CBD) cycle of two miles with seven stops per mile and a top speed of 20 mph is shown in Fig. D-1. The Arterial Route Cycle of two miles with two stops per mile and a top speed of 40 mph is shown in Fig. D-2. The tests were actually run with top speeds of 15 mph and 35 mph, respectively. The minimum acceleration curve from the White Book is shown in Fig. D-3.

**Table D-1 Hybrid Electric Transit Bus Test Matrix**

PARAMETER	CAP & APU REGEN	CAP & APU NO REGEN	BAT & APU REGEN	BAT & APU NO REGEN
Central Business District Cycle (20 mph max, 15 mph max as run).	Y	Y	Y	Y
Arterial Route Cycle (40 mph max , 35 mph max as run)	N	N	Y	Y
Top Speed (60 mph min with all accessories operating).	N	N	Y	Y
Gradeability (34 mph on a 2.5% grade, 7 mph on a 12% grade with all accessories operating).	Y	Y	Y	Y
Acceleration (0.06 g average between 0 and 15 mph).	Y	Y	Y	Y
Jerk (0.3 g max, accelerating and decelerating).	Y	Y	Y	Y
Sound Level (Central Business District Cycle)	Y	Y	Y	Y
Sound Level (Constant 20 mph).	Y	Y	Y	Y
Fuel Economy (Central Business District - see below).	Y	Y	N	N

**Table D-2 Transit Coach Design Operating Duty Cycle**

PHASE	STOPS	TOP SPEED (MPH)	MILES	ACC DIST (FT)	ACC TIME (SEC)	CRUISE DIST (FT)	CRUISE TIME (SEC)	DEC RATE (FT/S/S)	DEC DIST (FT)	DEC TIME (SEC)	DWELL TIME (SEC)	CYC TIME (M-S)	TOT STOPS
CBD	7	20	2	155	10	540	18.5	6.78	60	4.5	7	9-20	14
IDLE												5-0	
ART	2	40	2	1035	29	1350	22.5	6.78	255	9	7	4-30	4
CBD	7	20	2	155	10	510	18.5	6.78	60	4.5	7	9-20	14
ART	2	40	2	1035	35	1350	22.5	6.78	255	9	7	4-30	4
CBD	7	20	2	155	10	510	18.5	6.78	60	4.5	7	9-20	14
COMM	1/PH	55	4	5500	90	15140	188	6.78	480	12	20	5-10	1
TOTAL			14									47-10	51

Fig. D-1

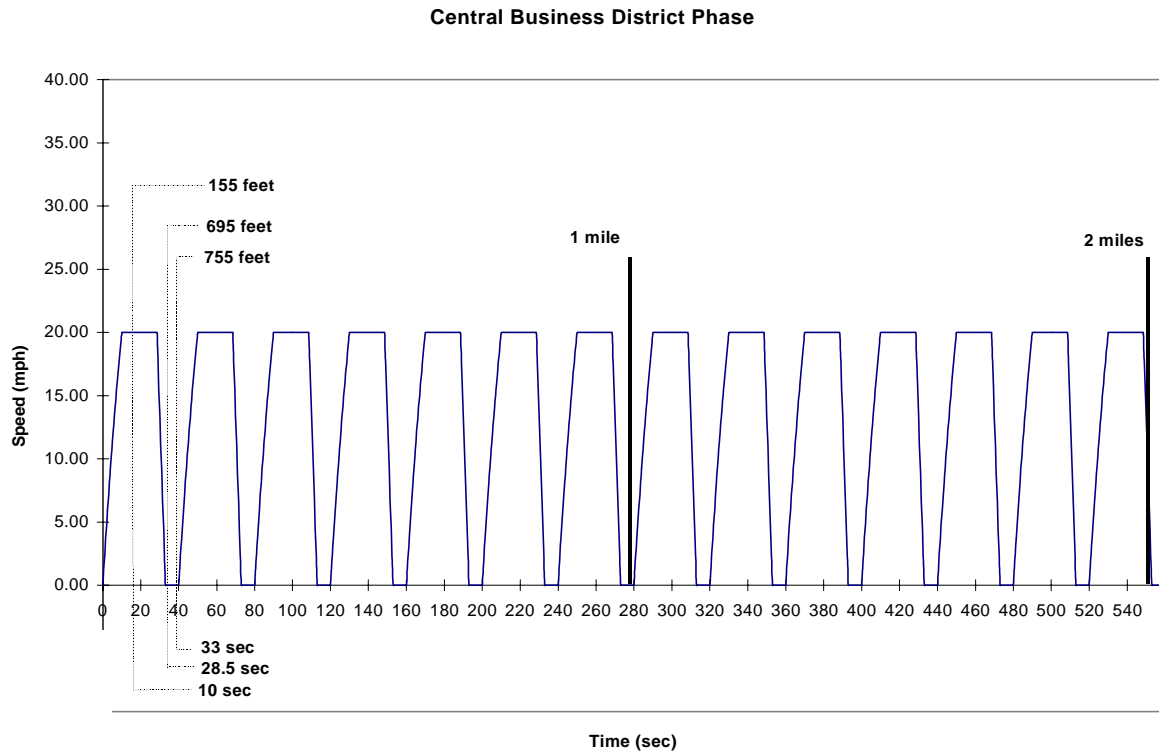


Fig. D-2

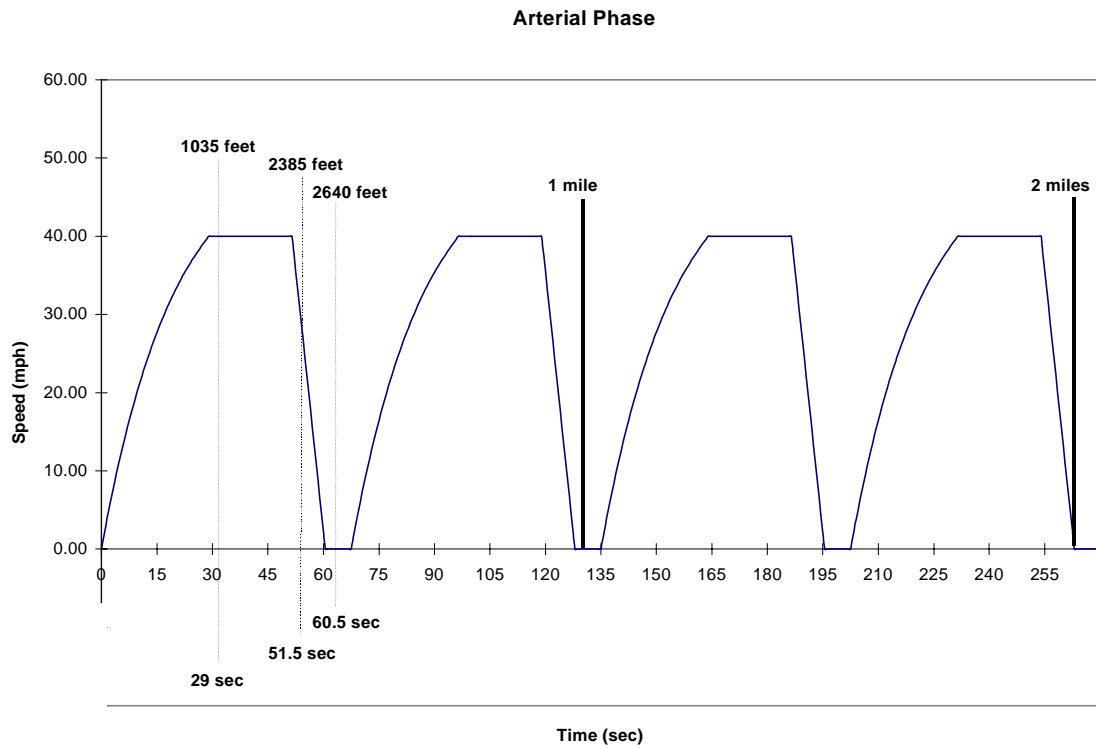
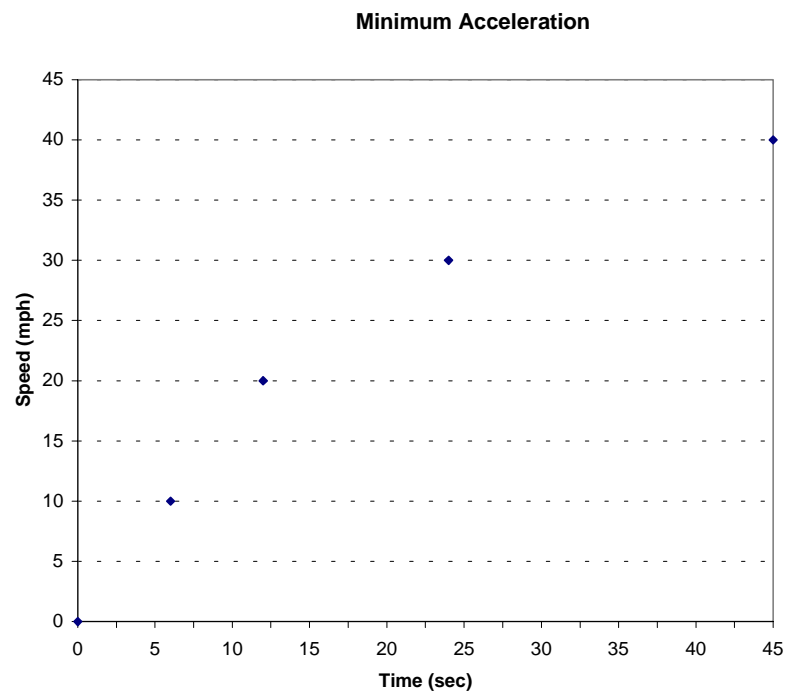


Fig. D-3

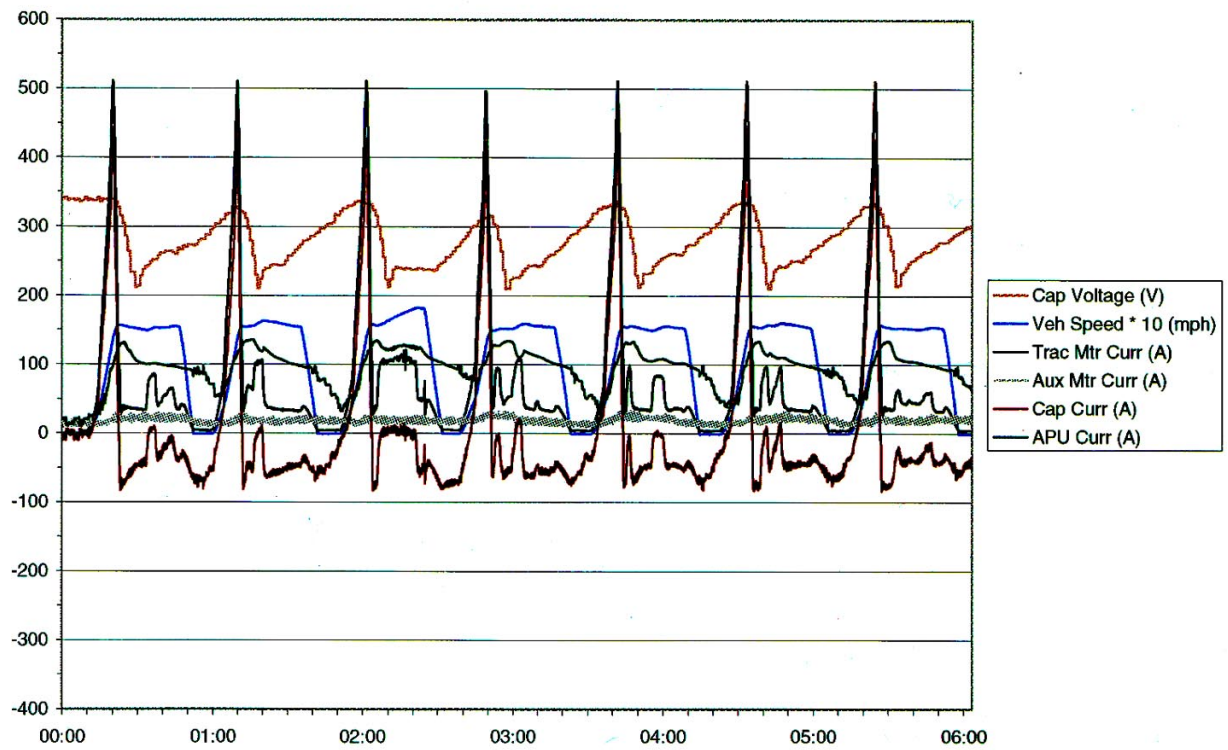


## APPENDIX E

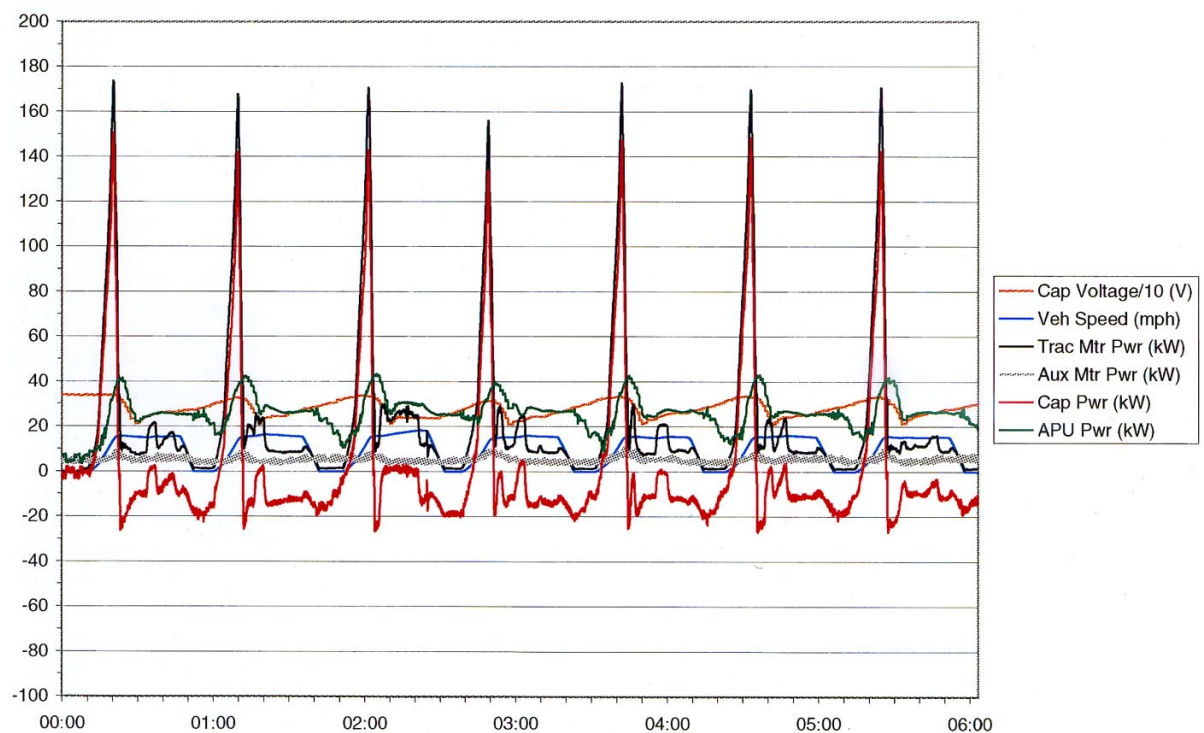
### VEHICLE PERFORMANCE TEST RESULTS

A complete set of plots of the test results are included here. Table 1 identifies the tests that were conducted.

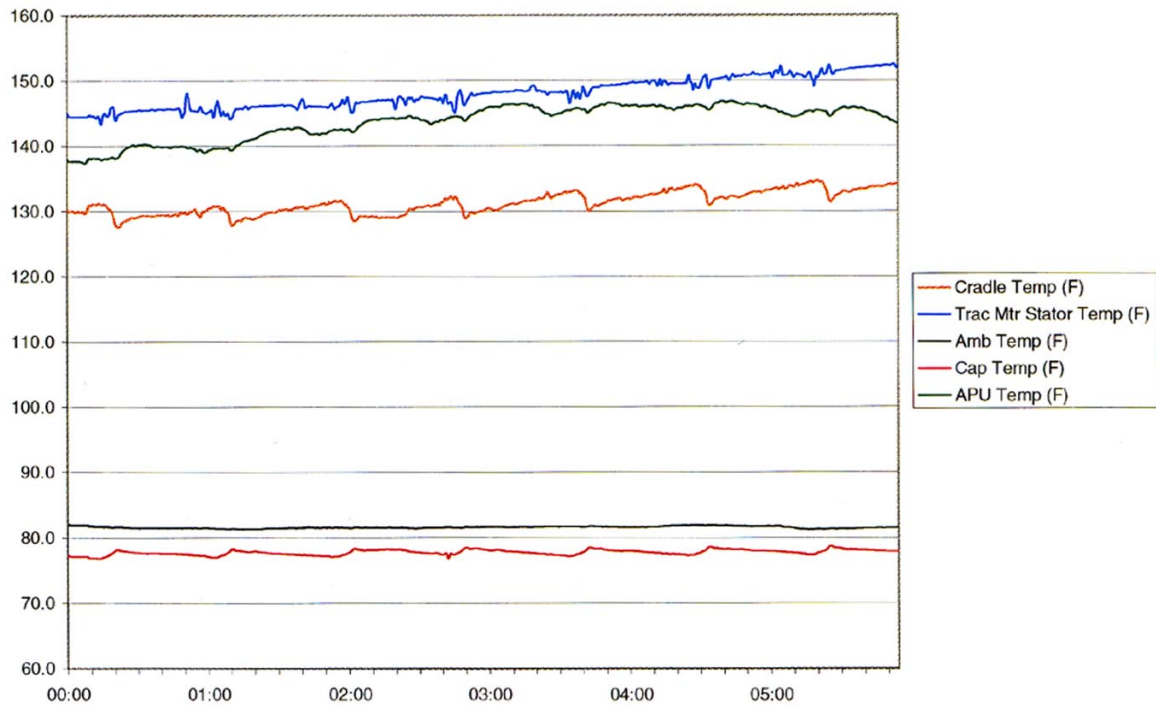
Test 1: Caps, CBD Fuel Economy, 15 mph, No Regen Braking



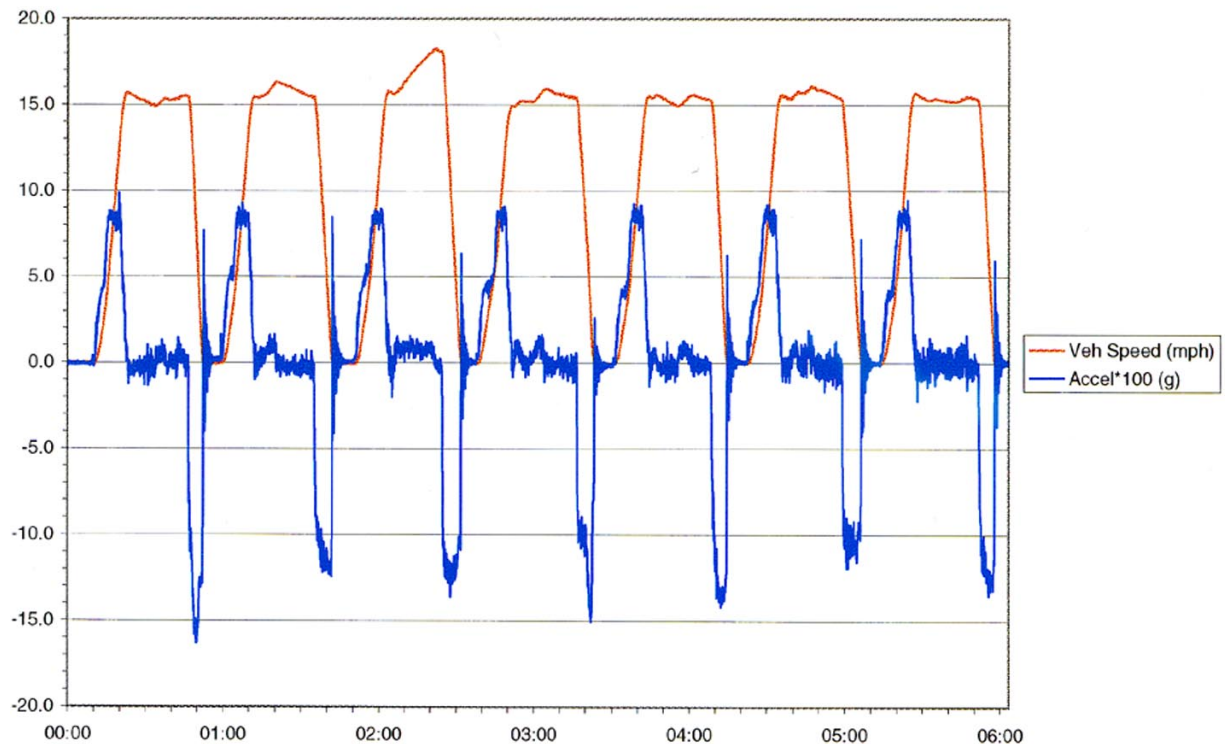
Test 1: Caps, CBD Fuel Economy, 15 mph, No Regen Braking



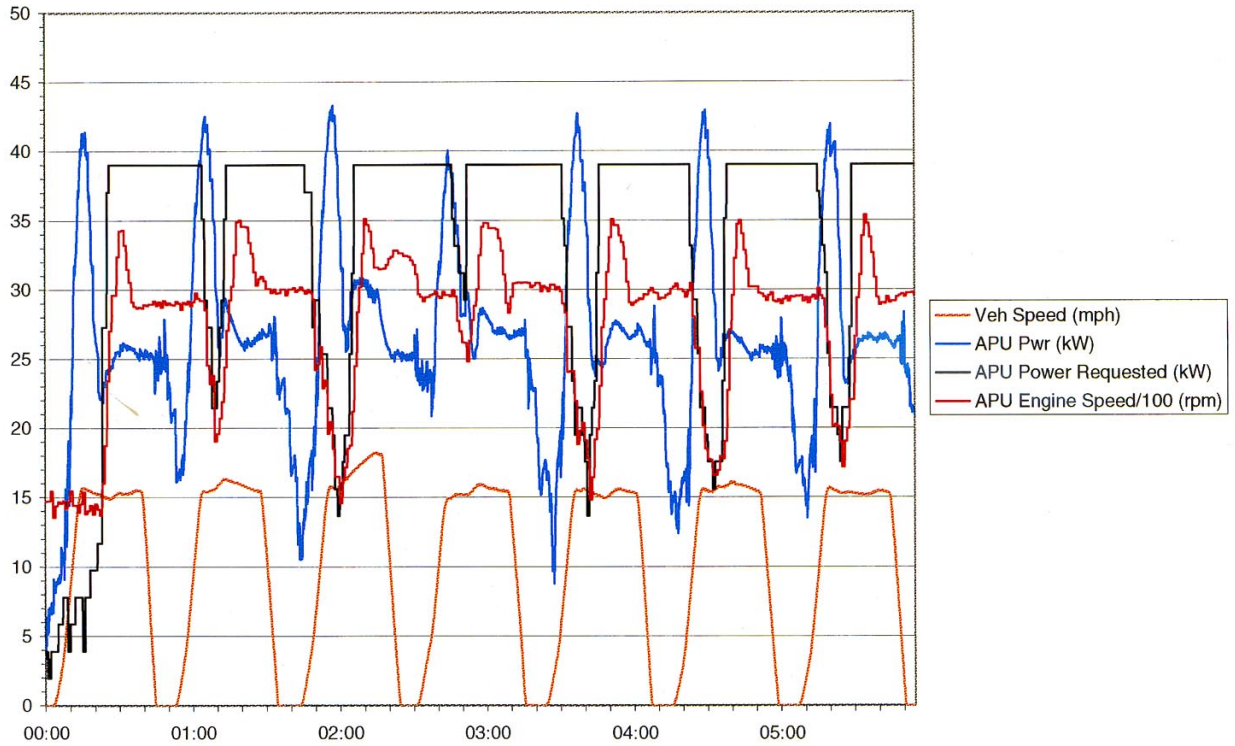
Test 1: Caps, CBD Fuel Economy, 15 mph, No RegenBraking



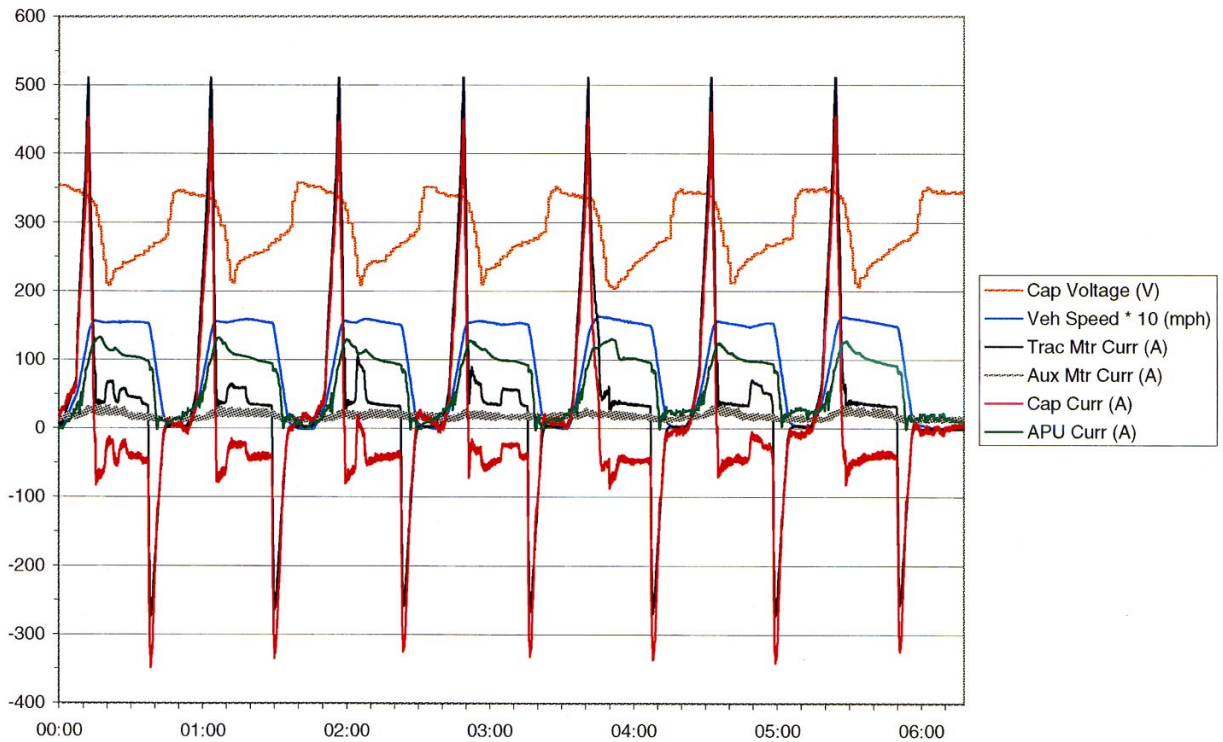
Test 1: Caps, CBD Fuel Economy, 15 mph, No Regen Braking



**Test 1: Capacitors, CBD Fuel Economy, 15 mph, No Regen Braking**

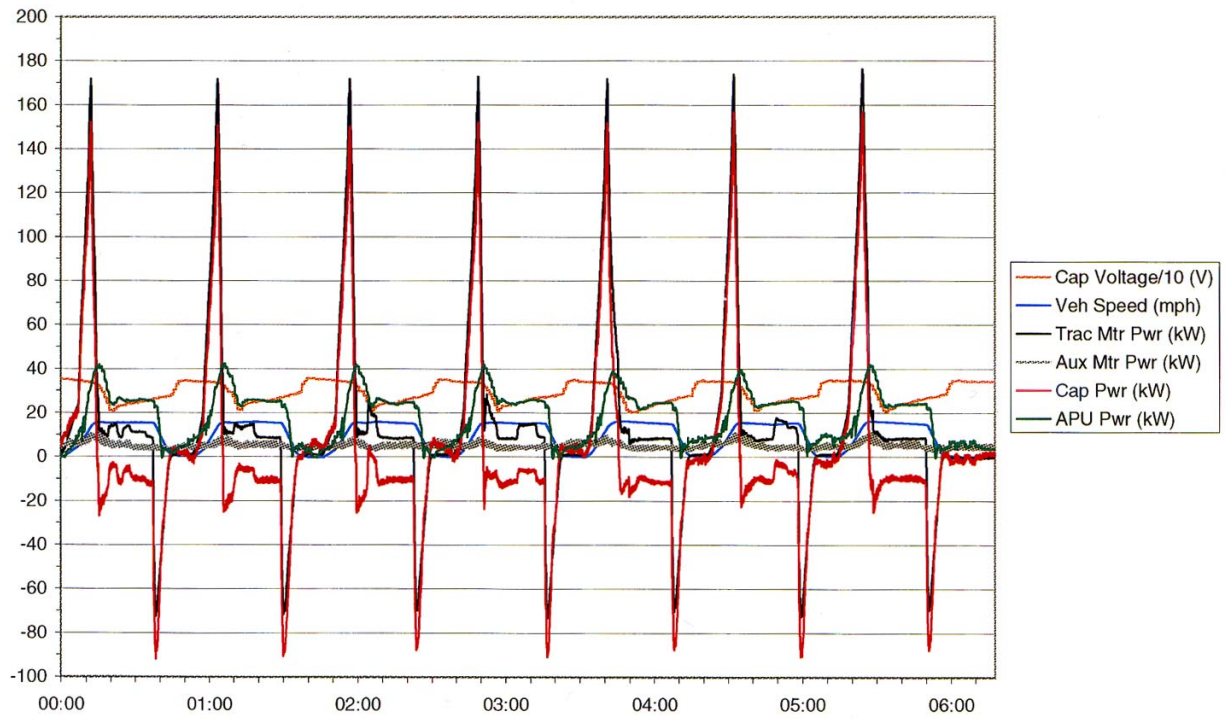


**Test 2: Capacitors, CBD Fuel Economy, 15 mph, Regen Braking**

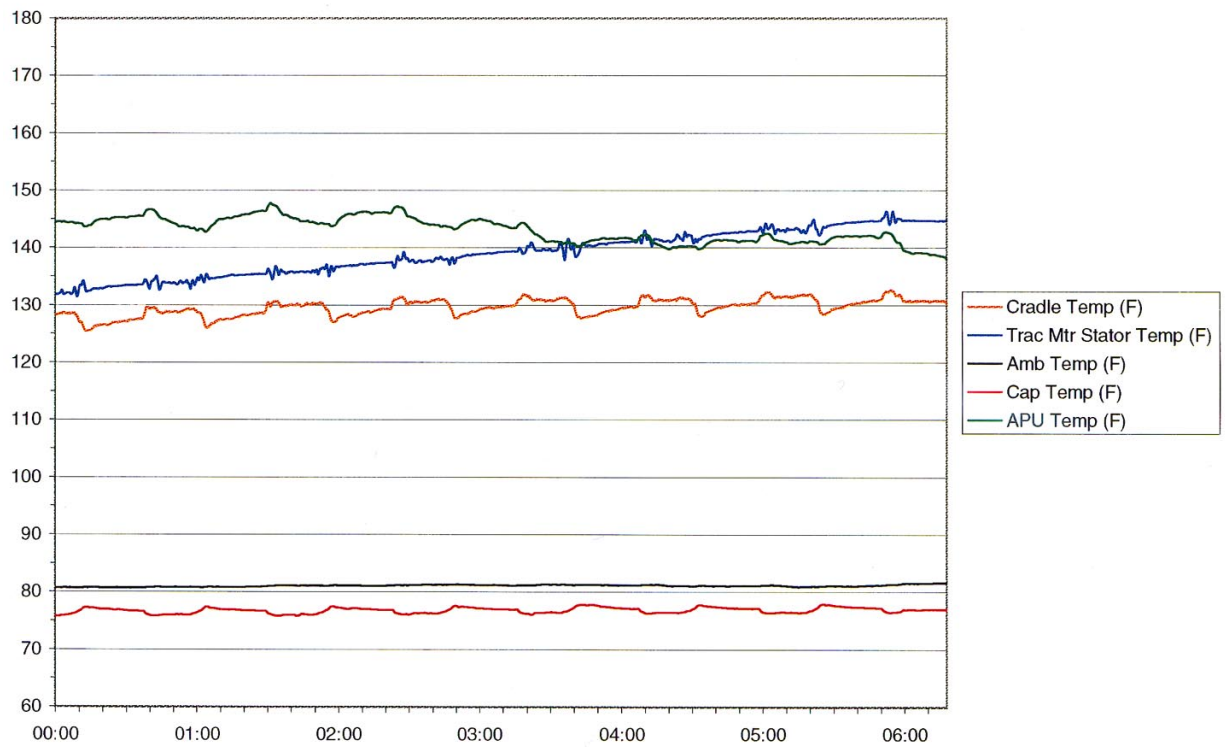




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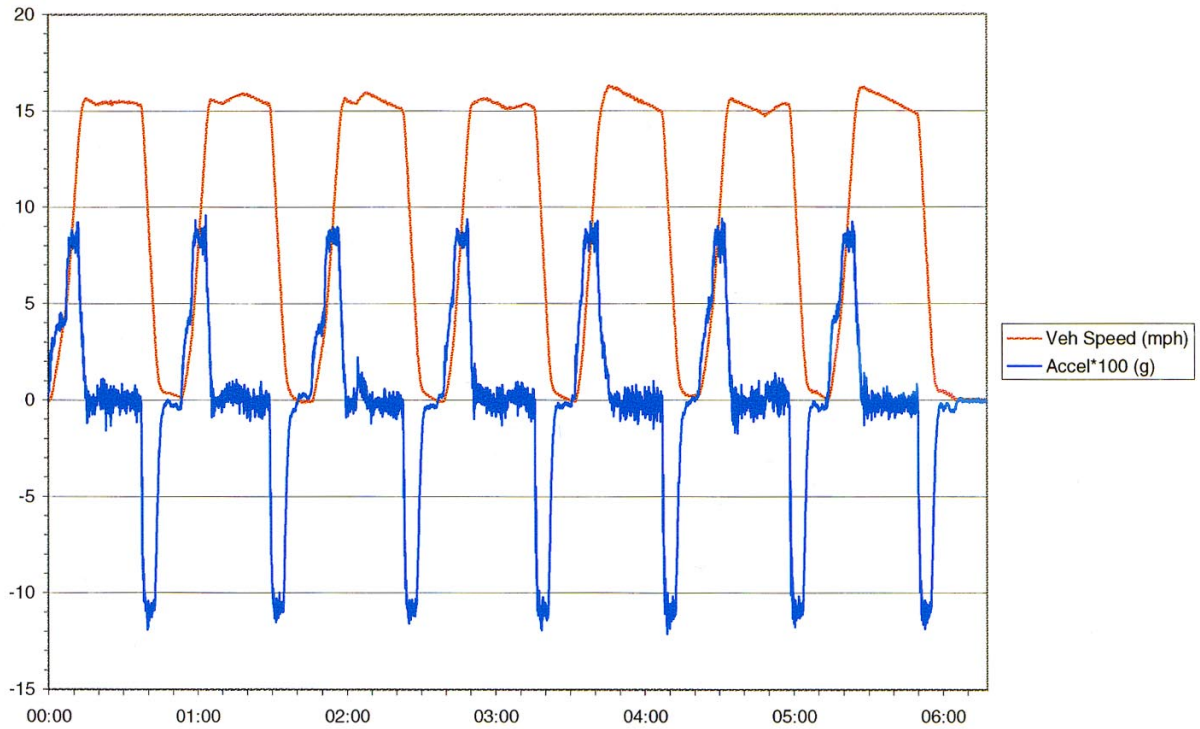


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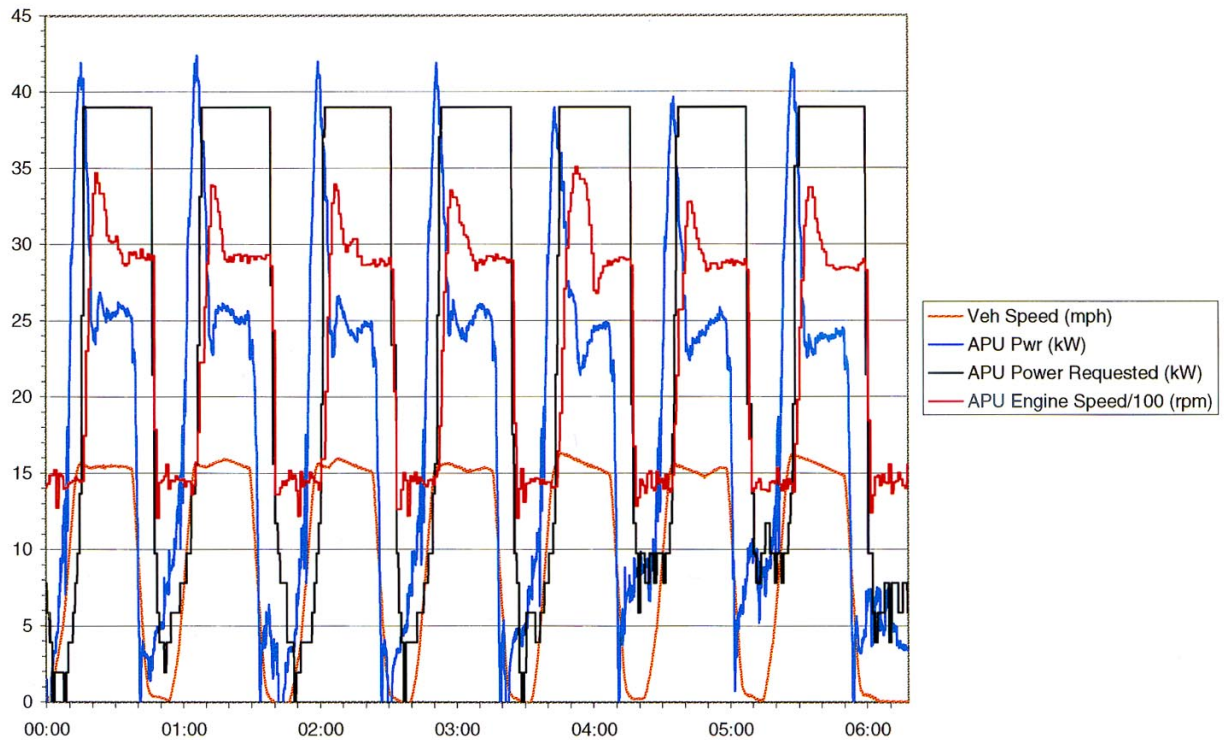




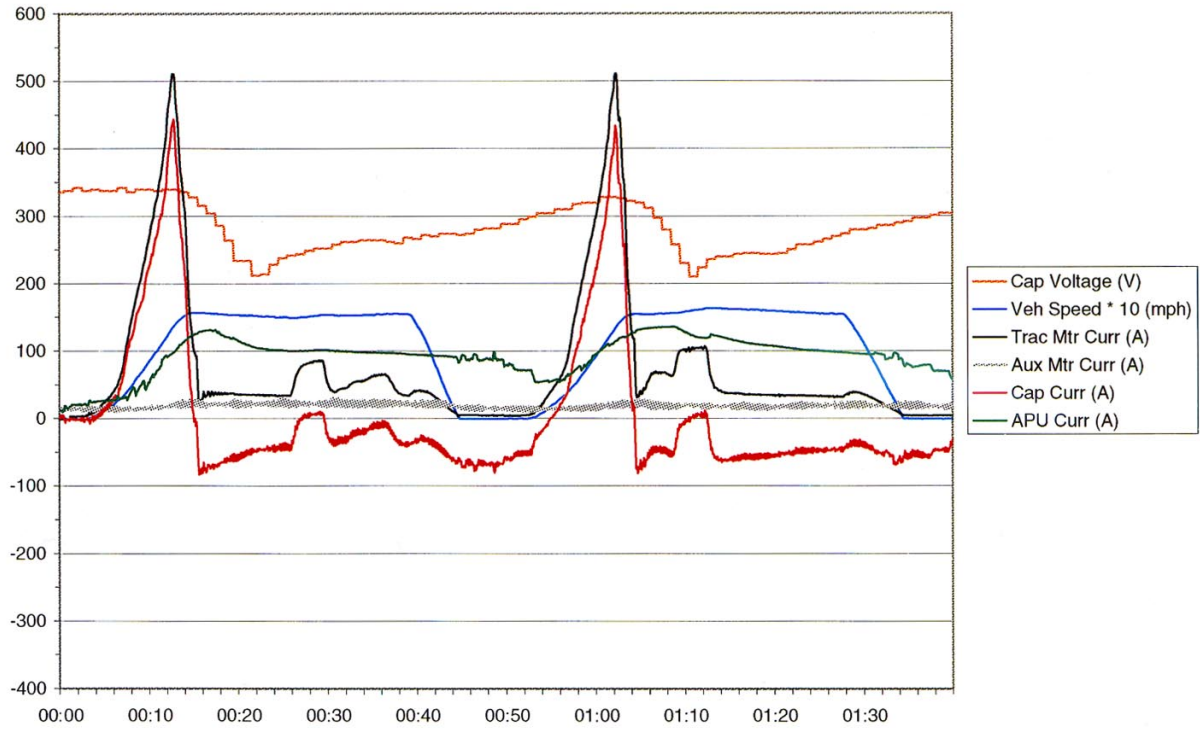
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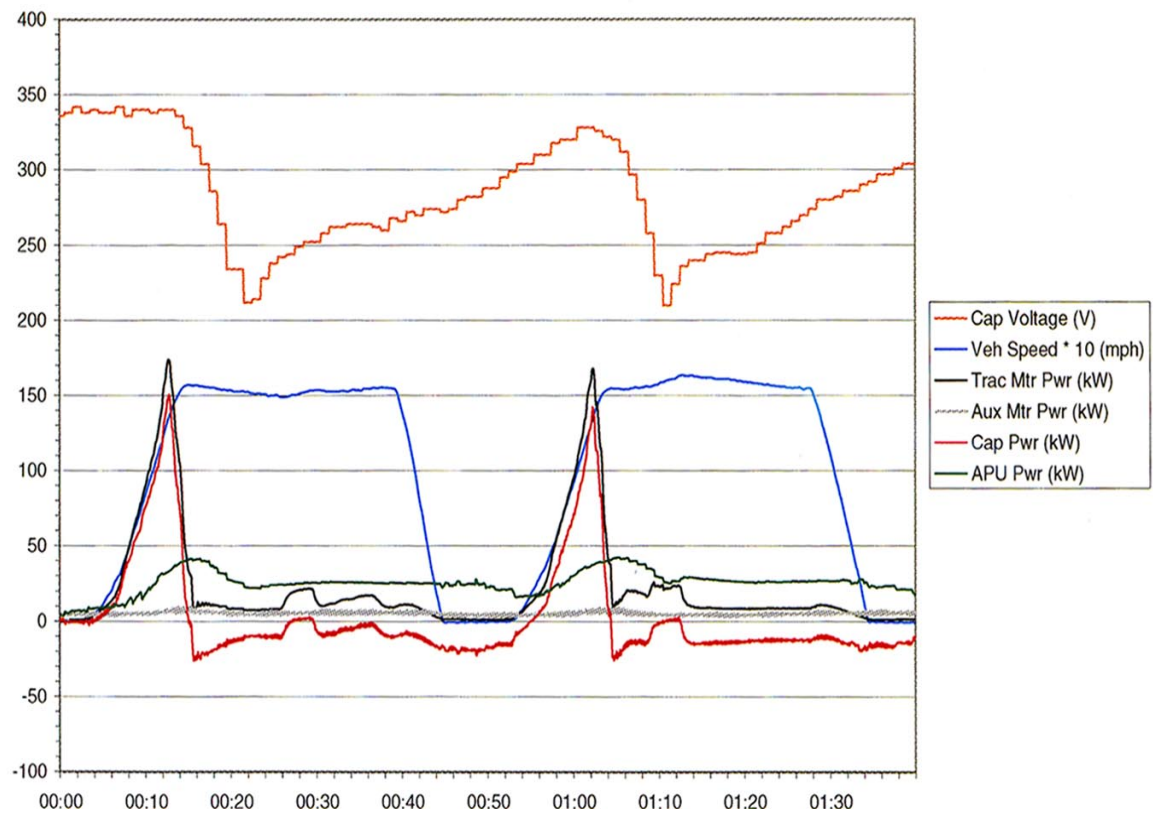
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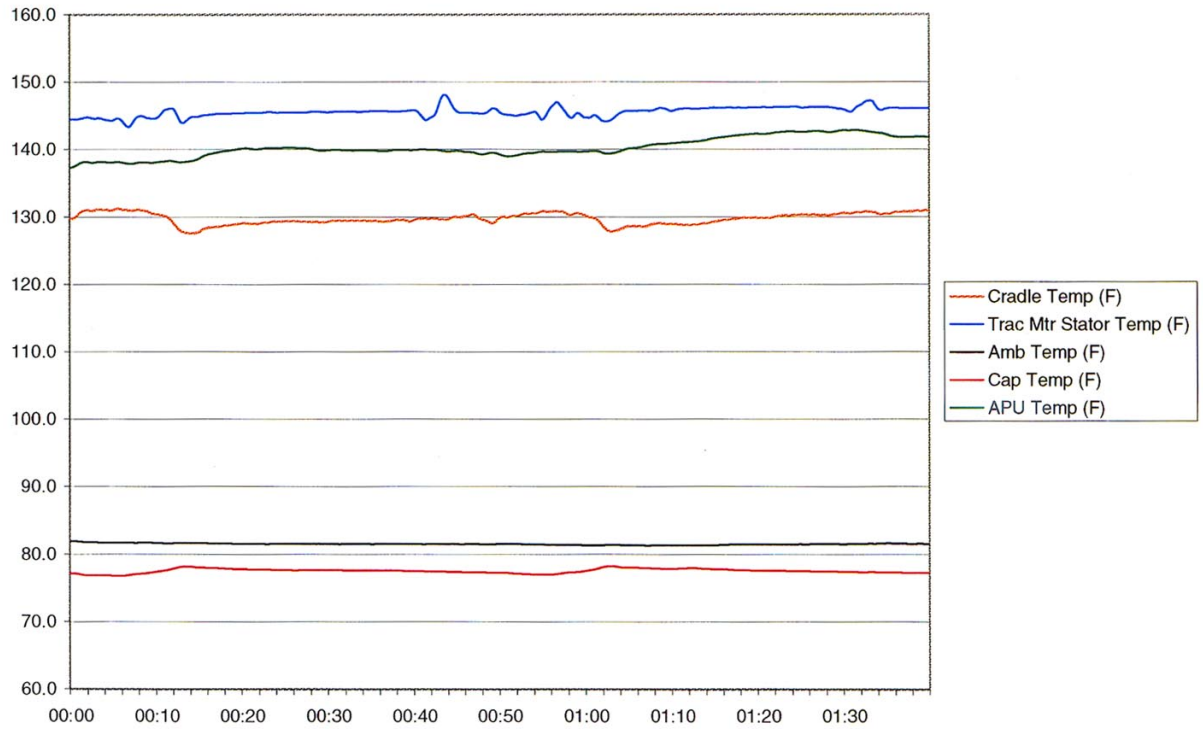
Test 3: Caps, 0-15 mph, No Regen Braking



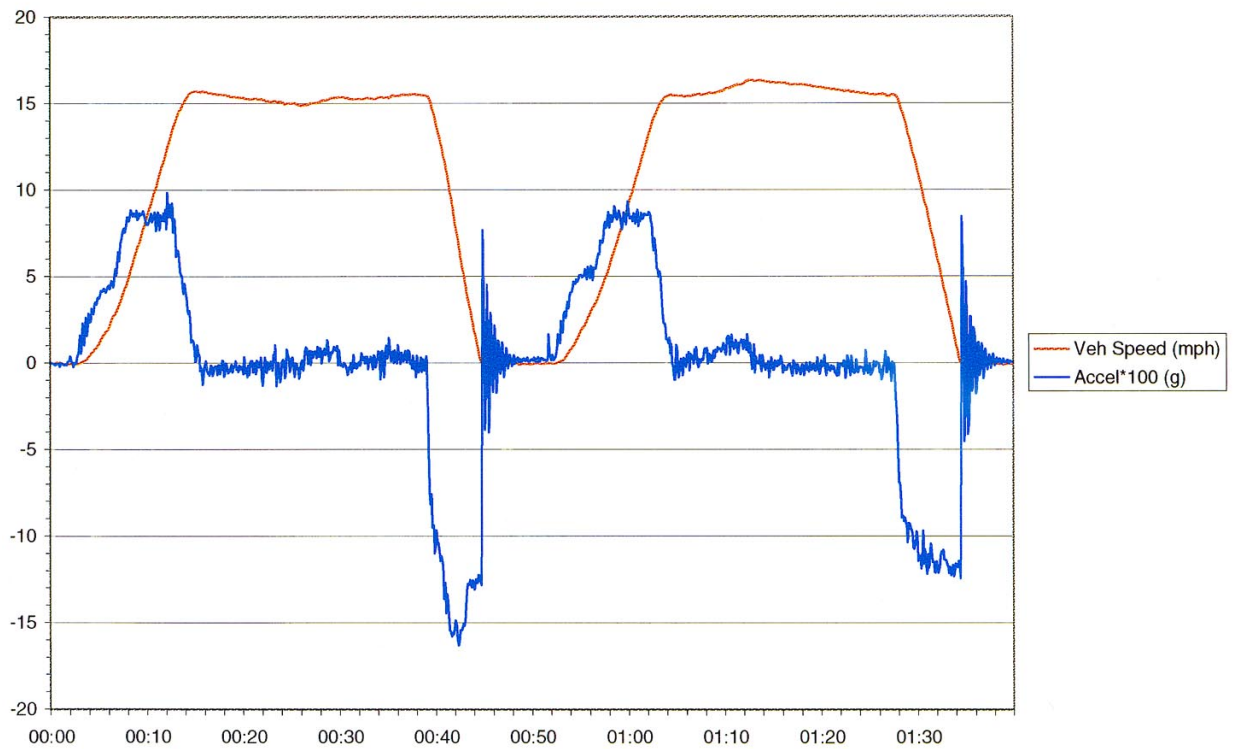
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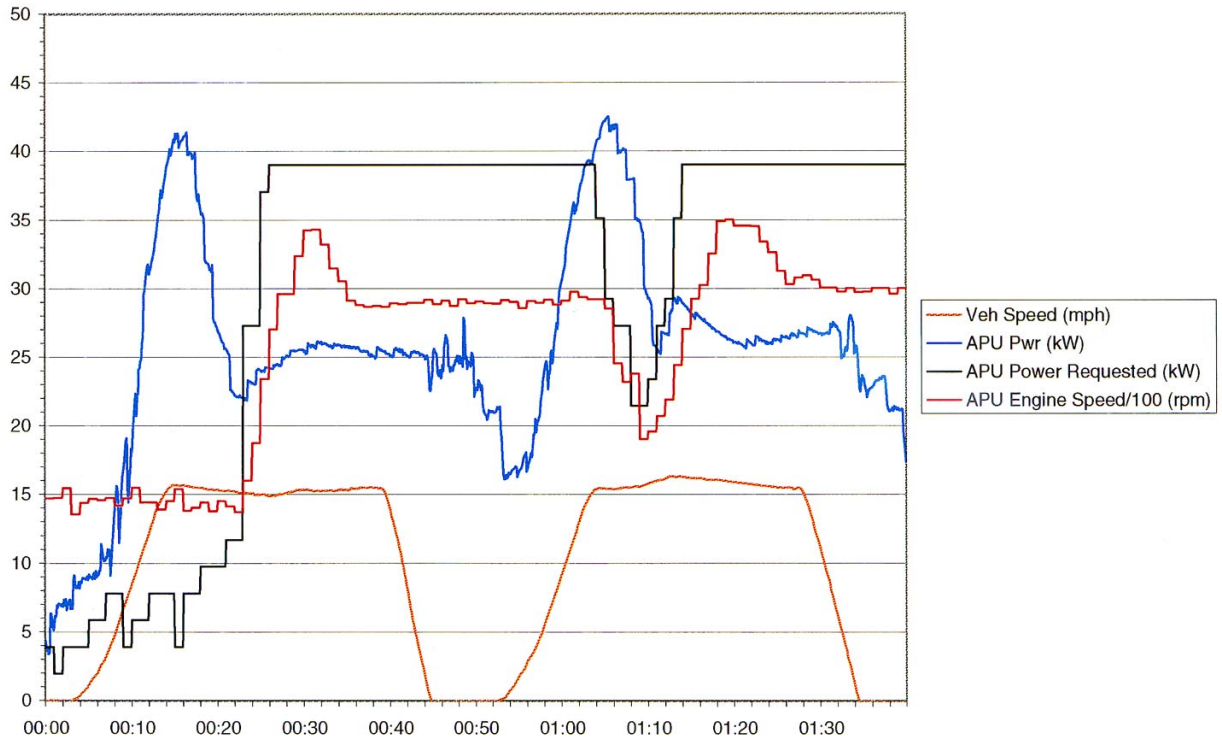
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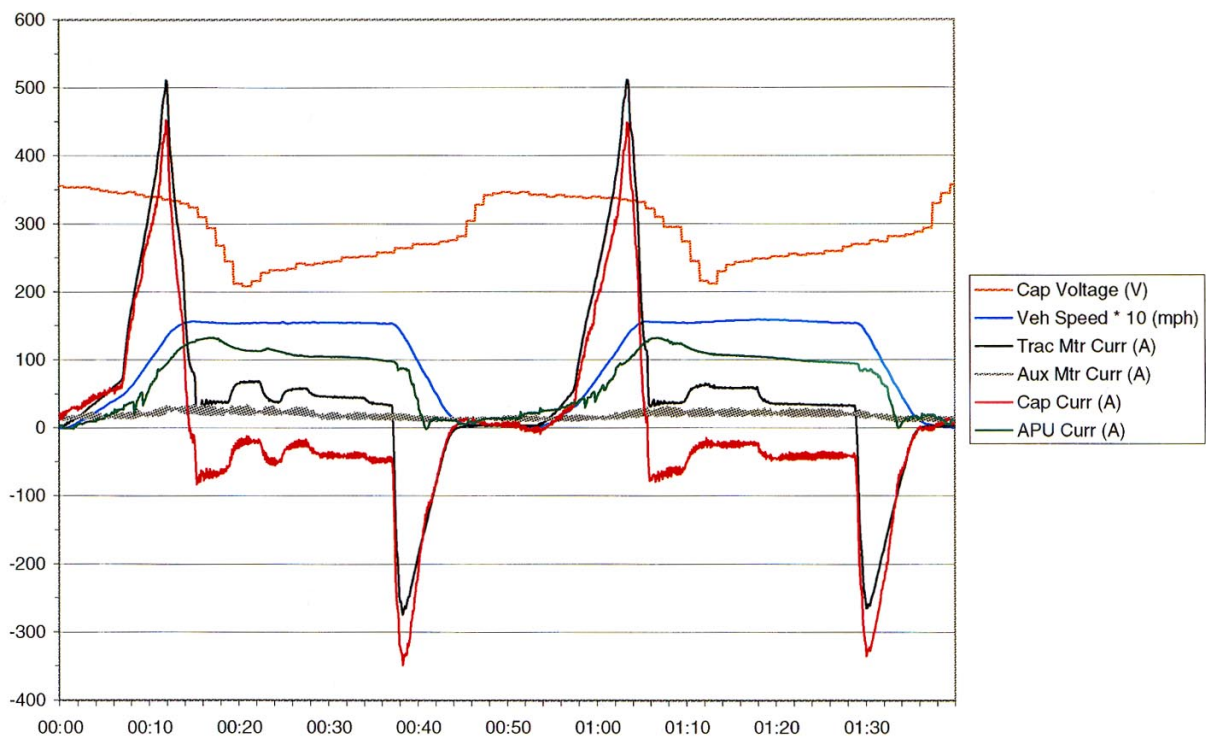
Test 3: Caps, 0-15 mph, No Regen Braking



**Test 3: Capacitors, 0-15 mph, No Regen Braking**

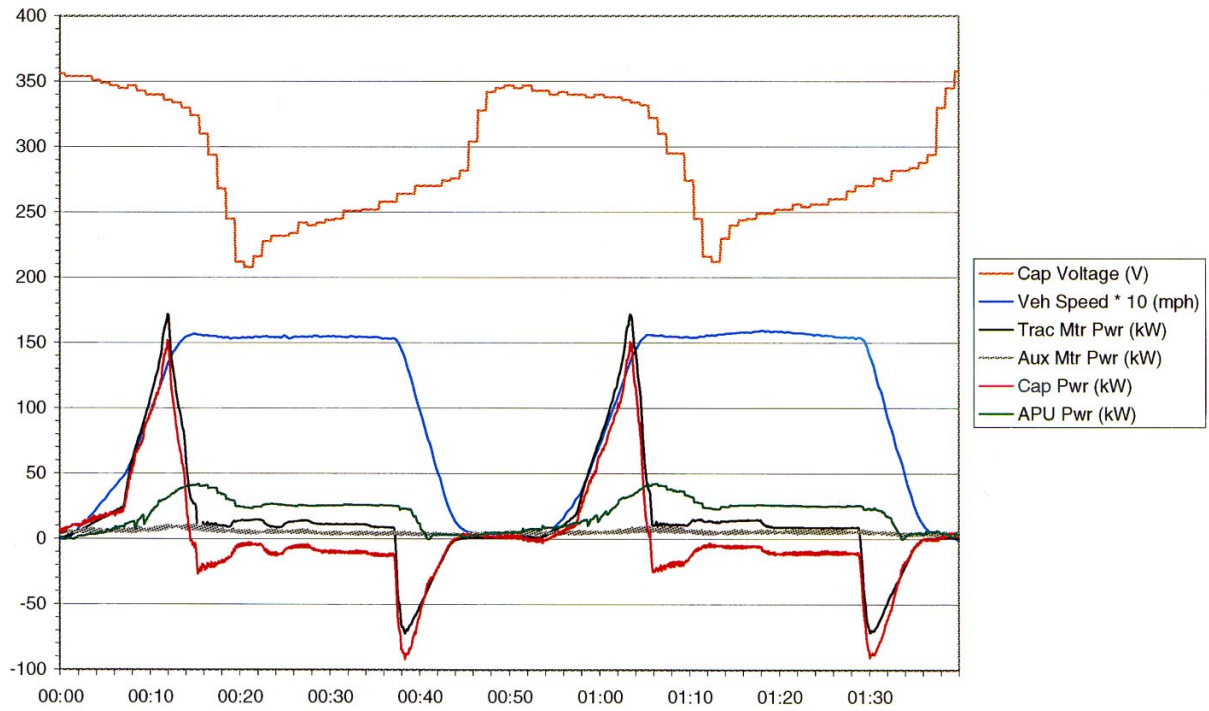


**Test 4: Capacitors, 0-15 mph, Regen Braking**

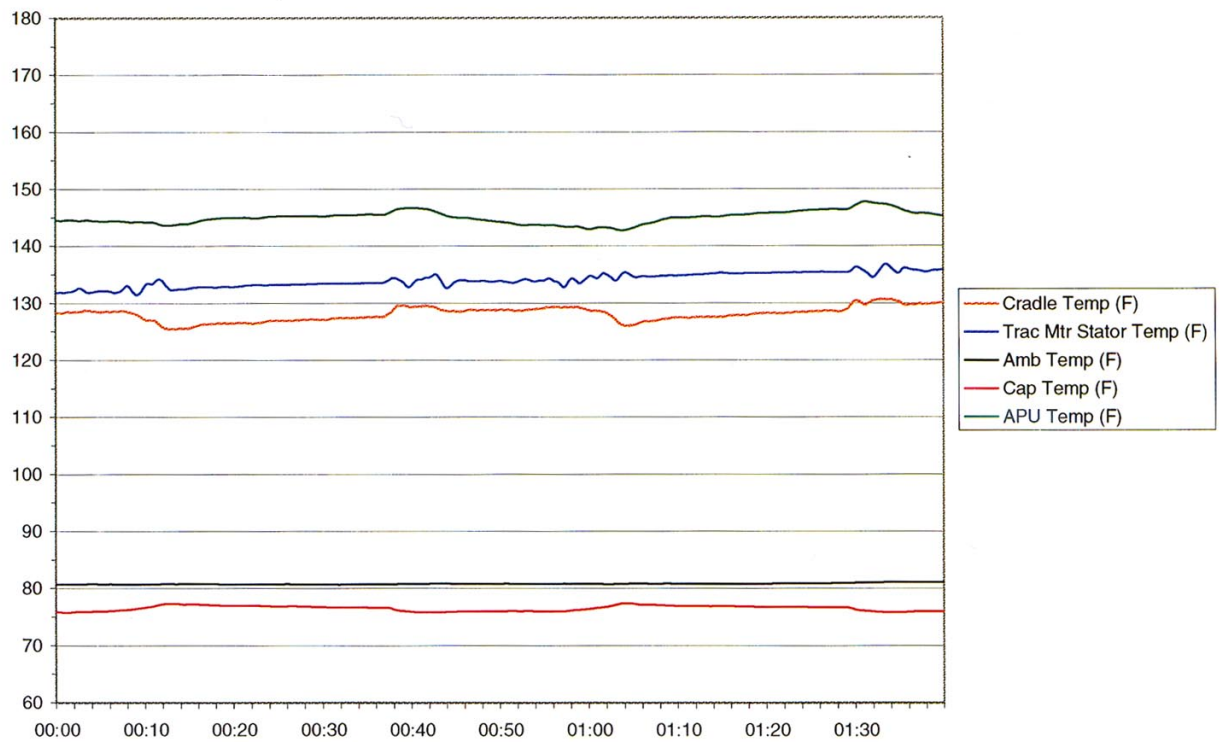




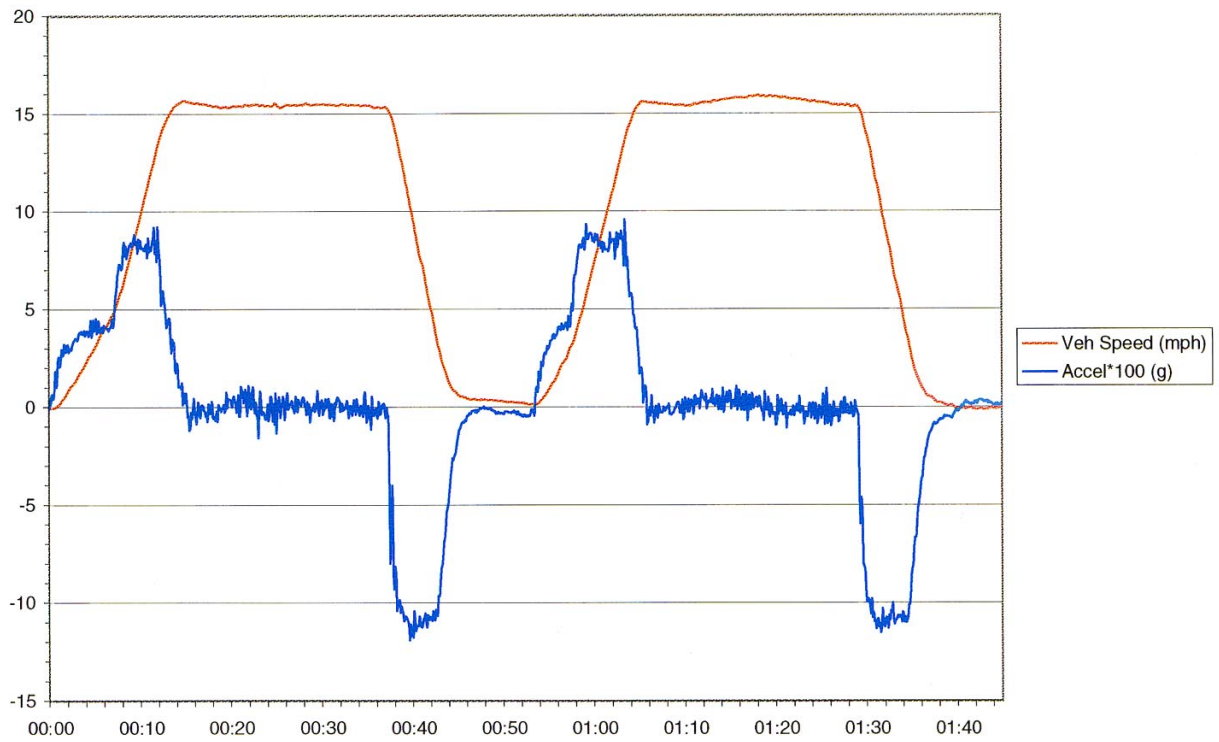
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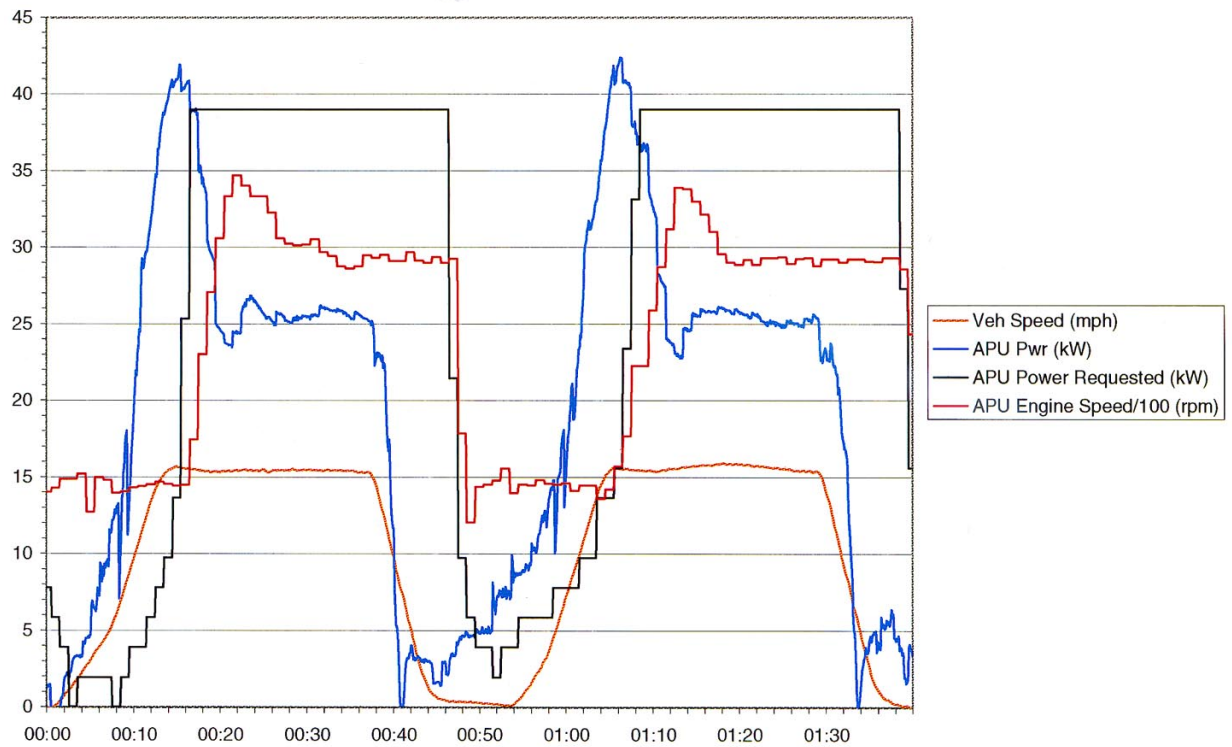
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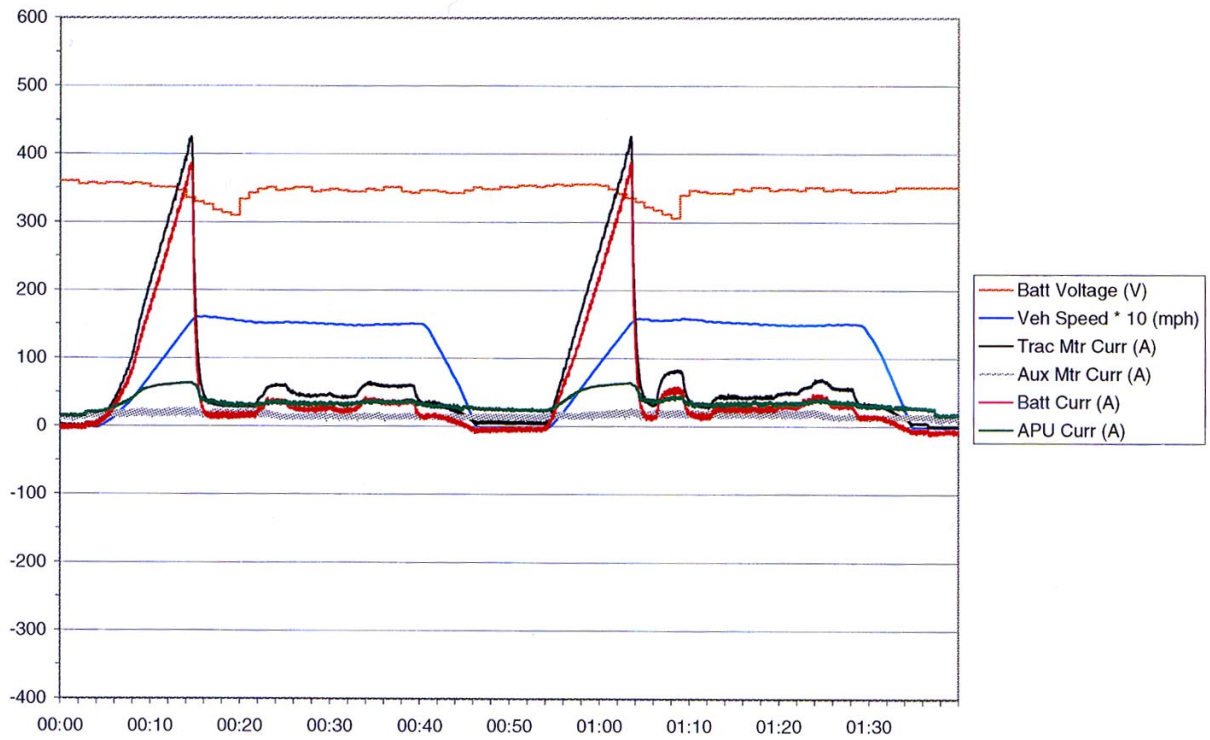
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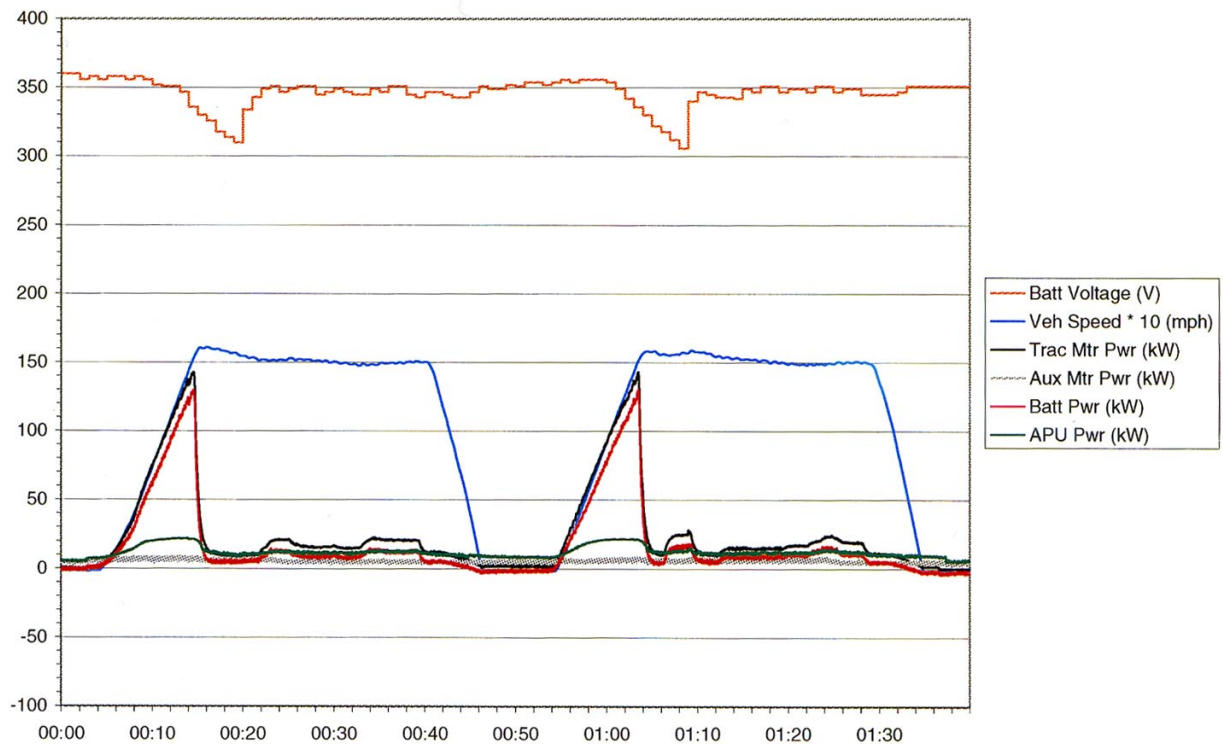
Test 4: Capacitors, 0-15 mph, Regen Braking



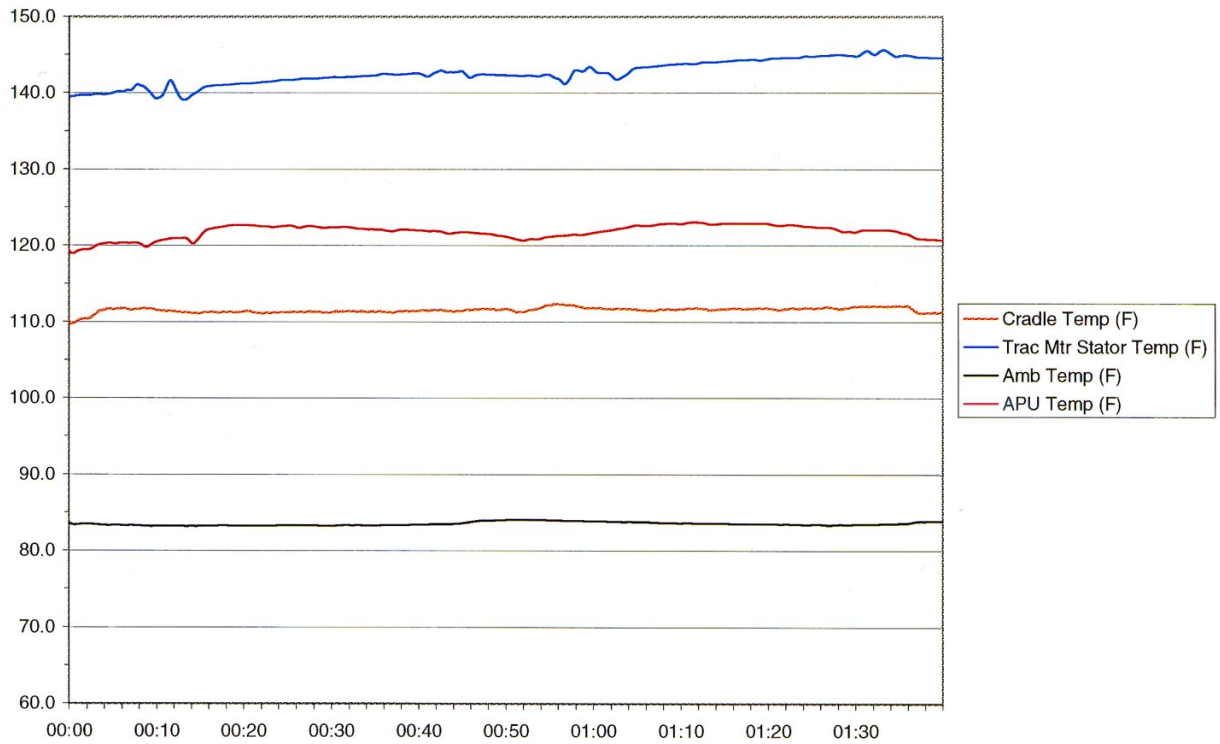
Test 5: Batteries, 0-15 mph, No Regen Braking



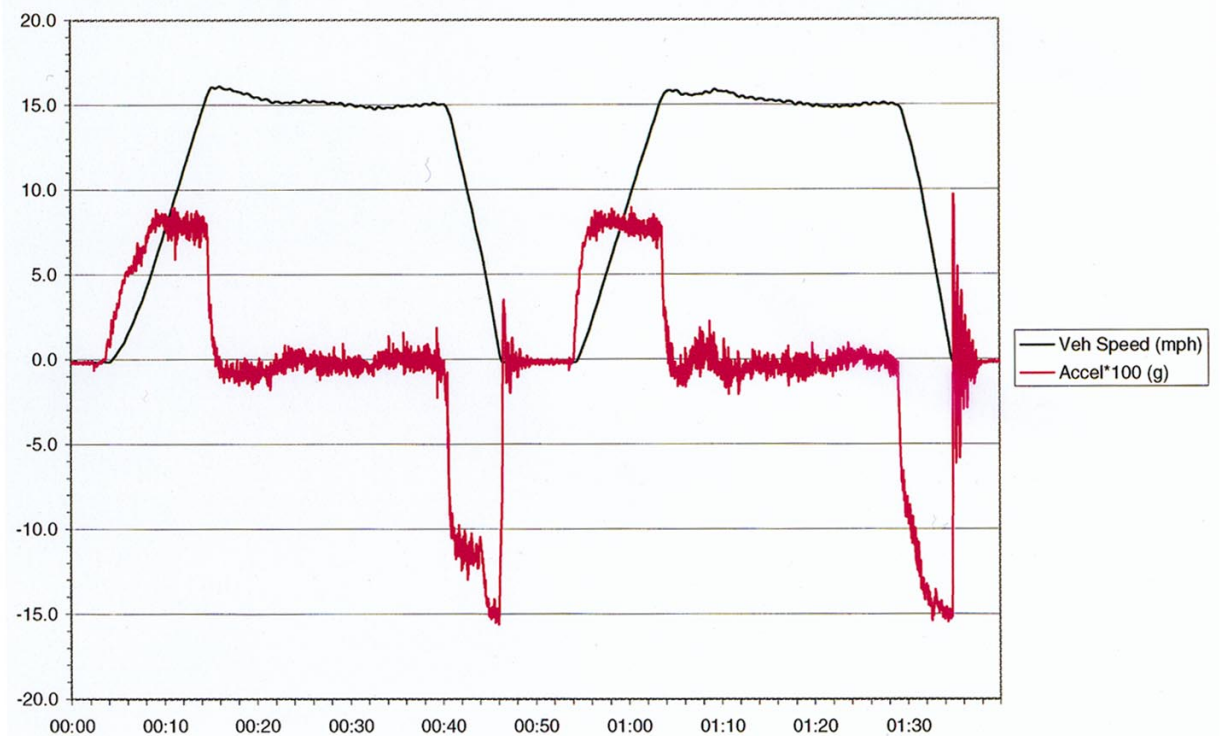
Test 5: Batteries, 0-15 mph, No Regen Braking



Test 5: Batteries, 0-15 mph, No Regen Braking

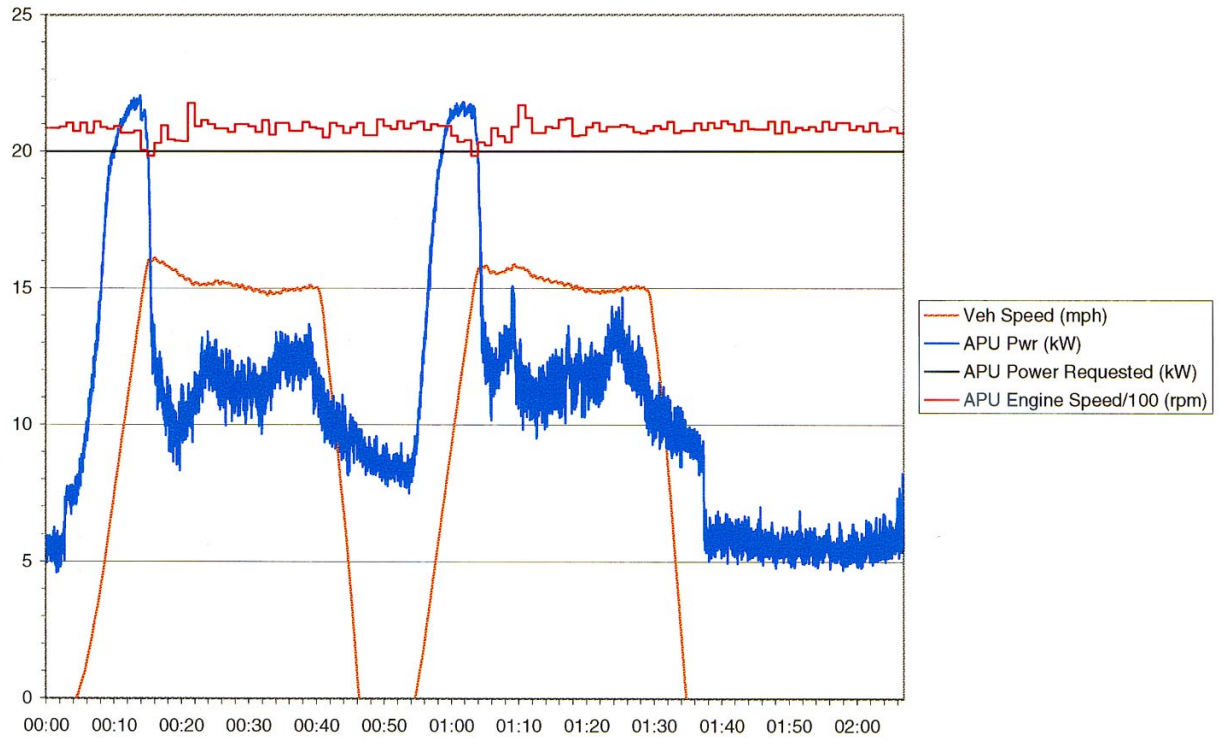


Test 5: Batteries, 0-15 mph, No Regen Braking

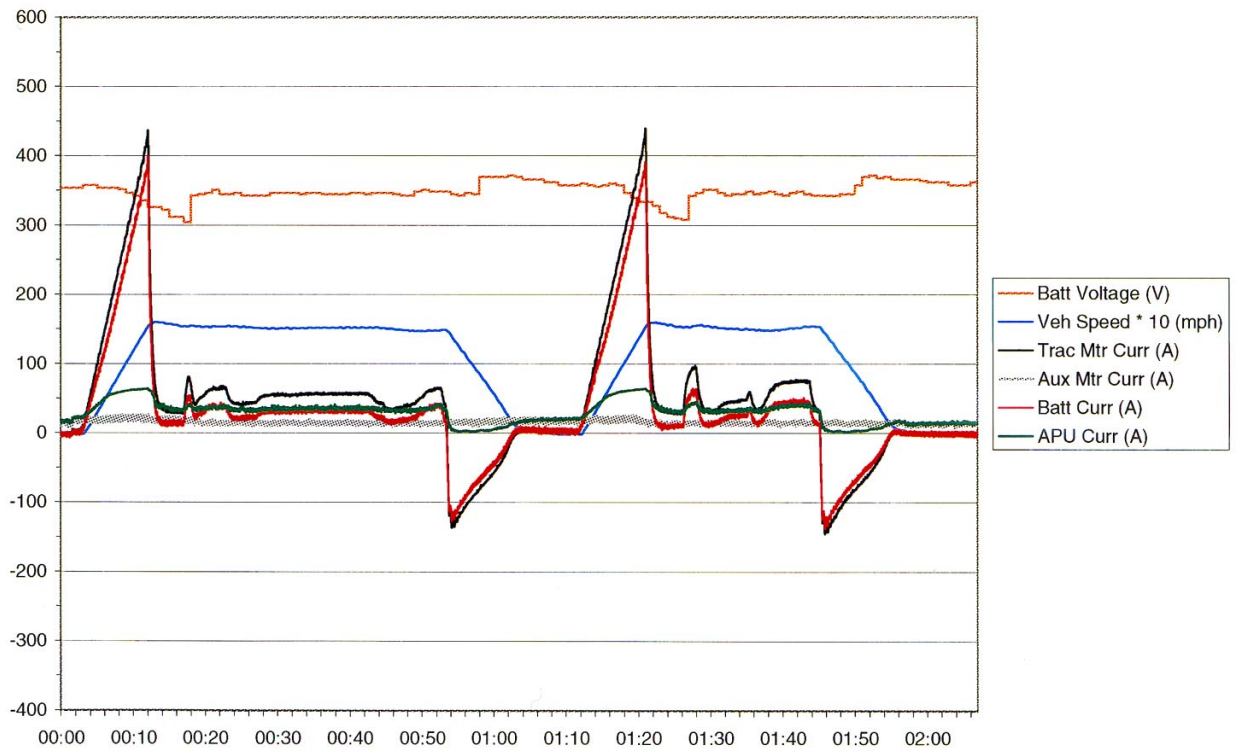




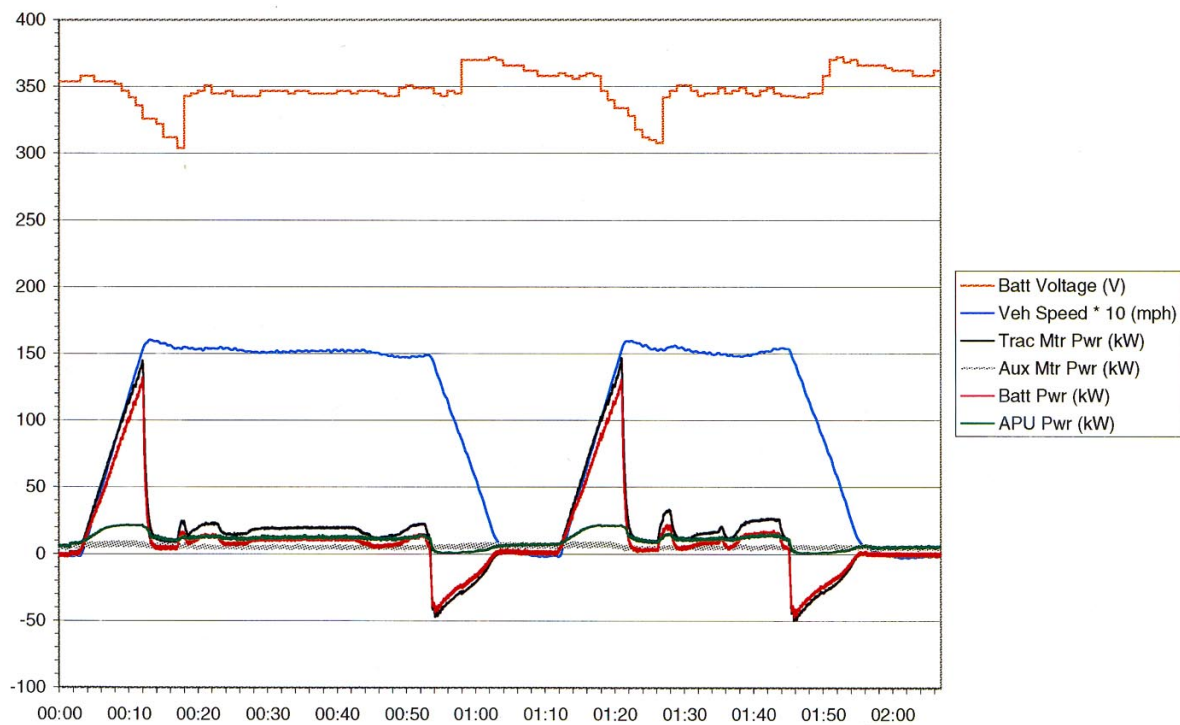
**Test 5: Batteries, 0-15 mph, No Regen Braking**



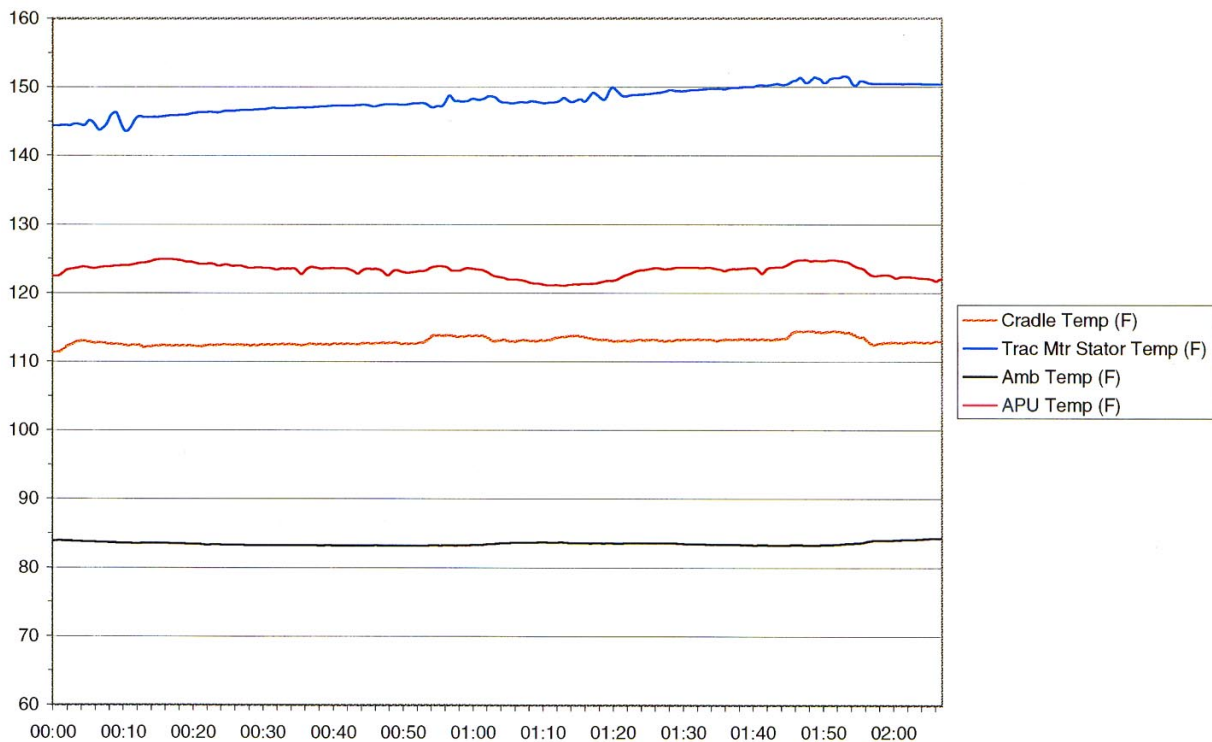
**Test 6: Batteries, 0-15 mph, Regen Braking**



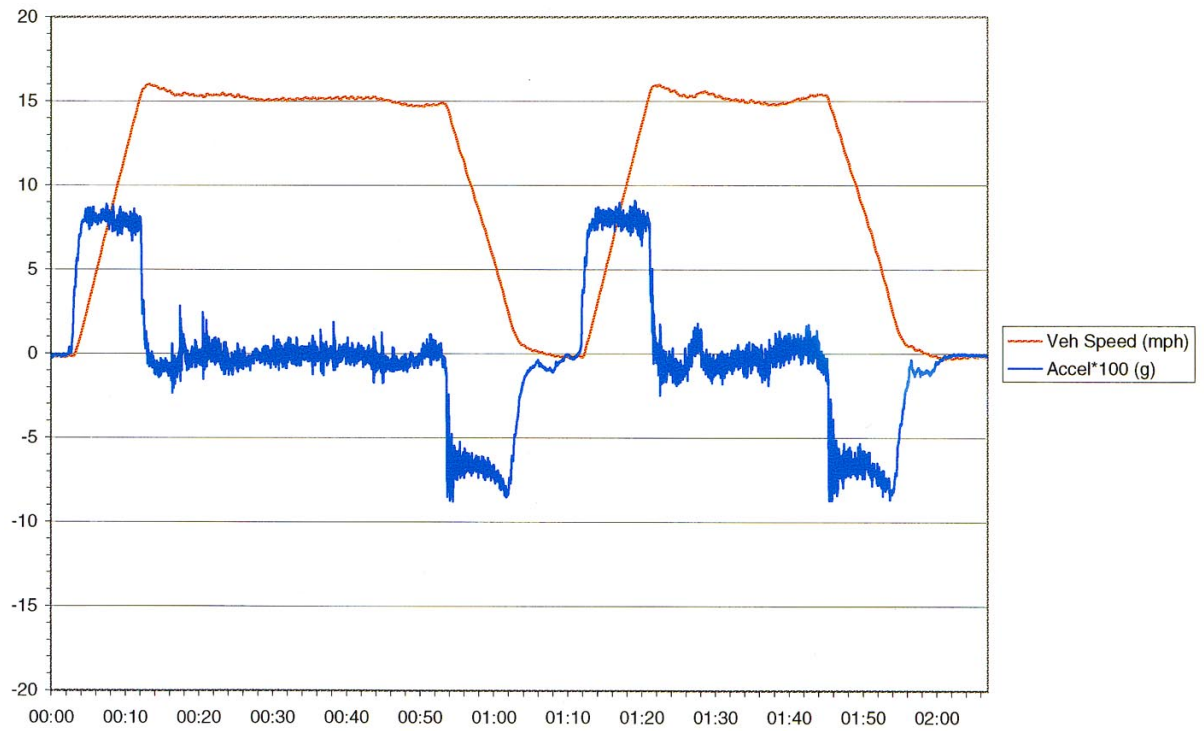
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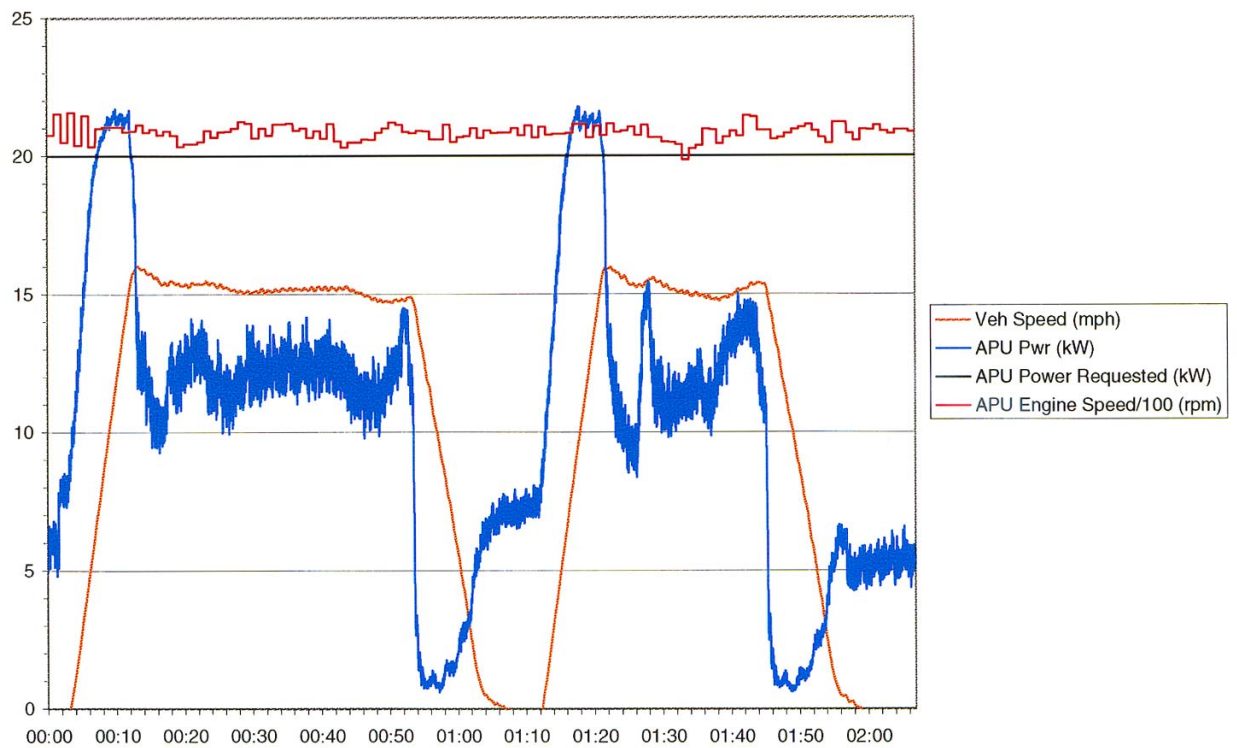
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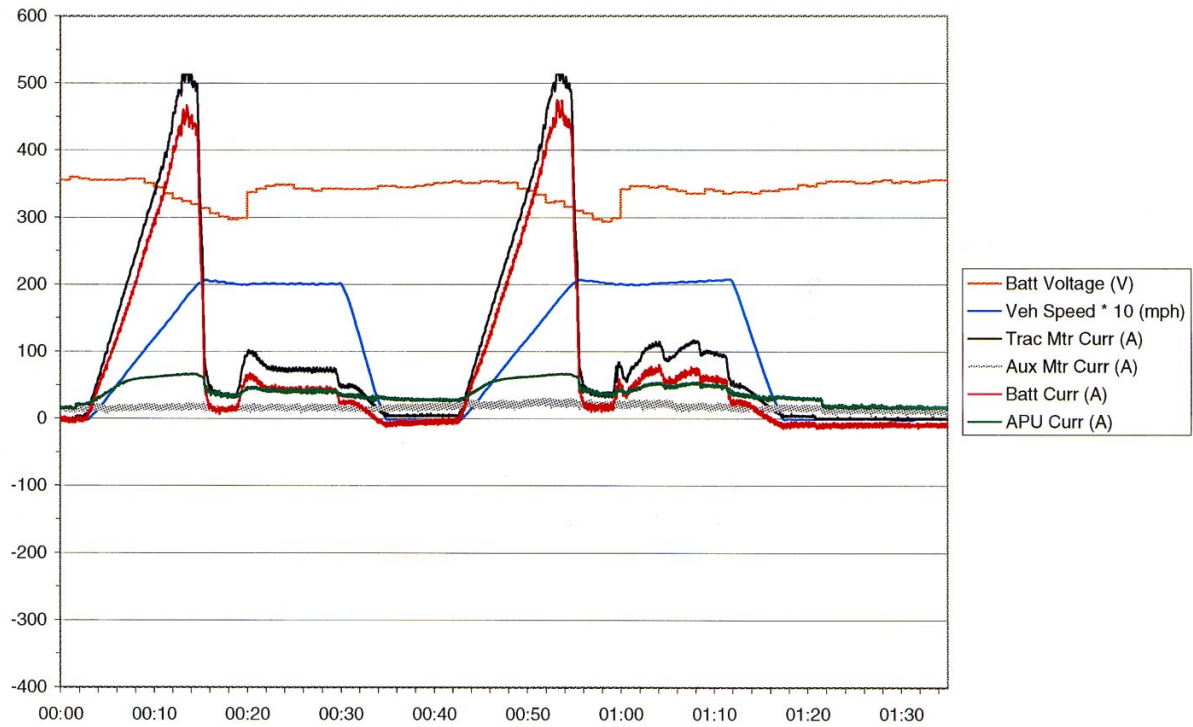
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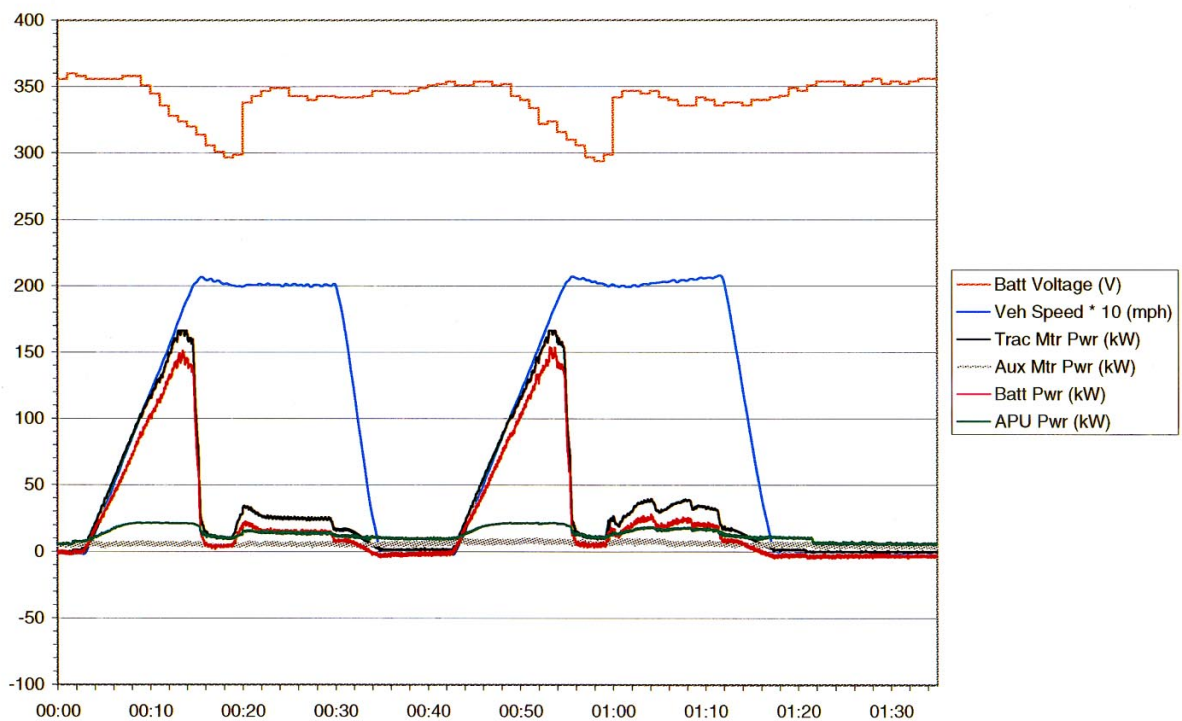
Test 6: Batteries, 0-15 mph, Regen Braking



Test 7: Batteries, 0-20 mph, No Regen Braking

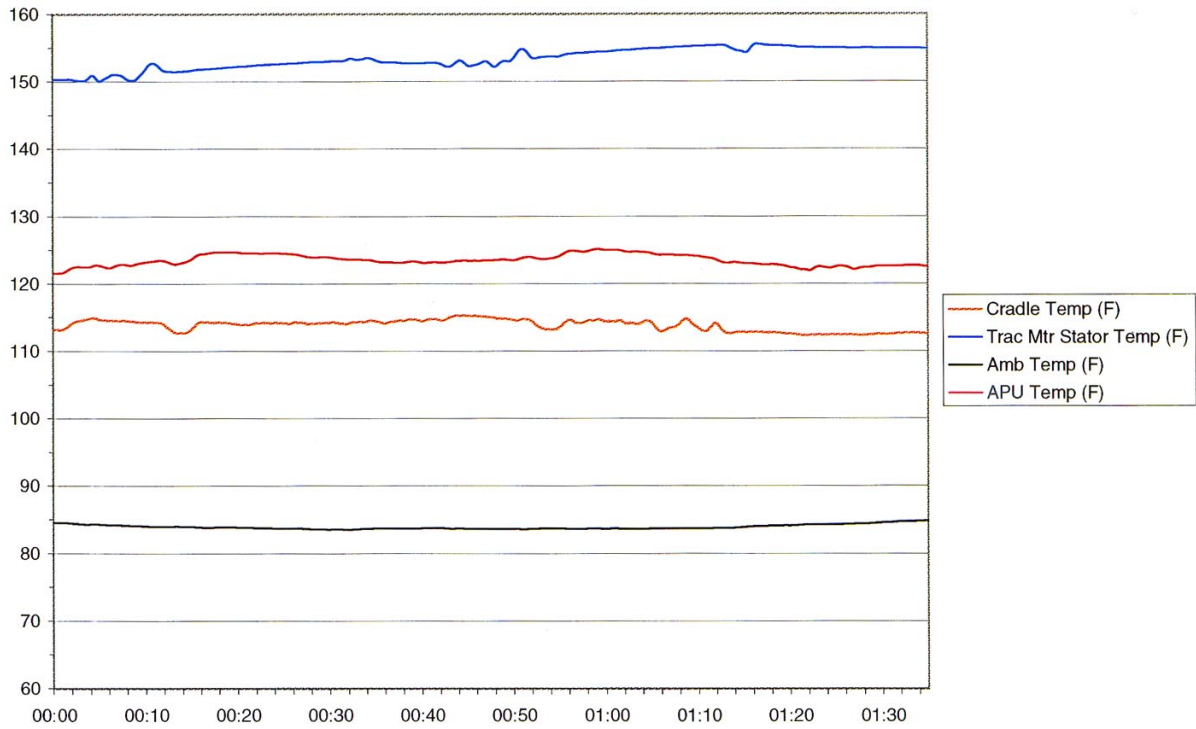


Test 7: Batteries, 0-20 mph, No Regen Braking

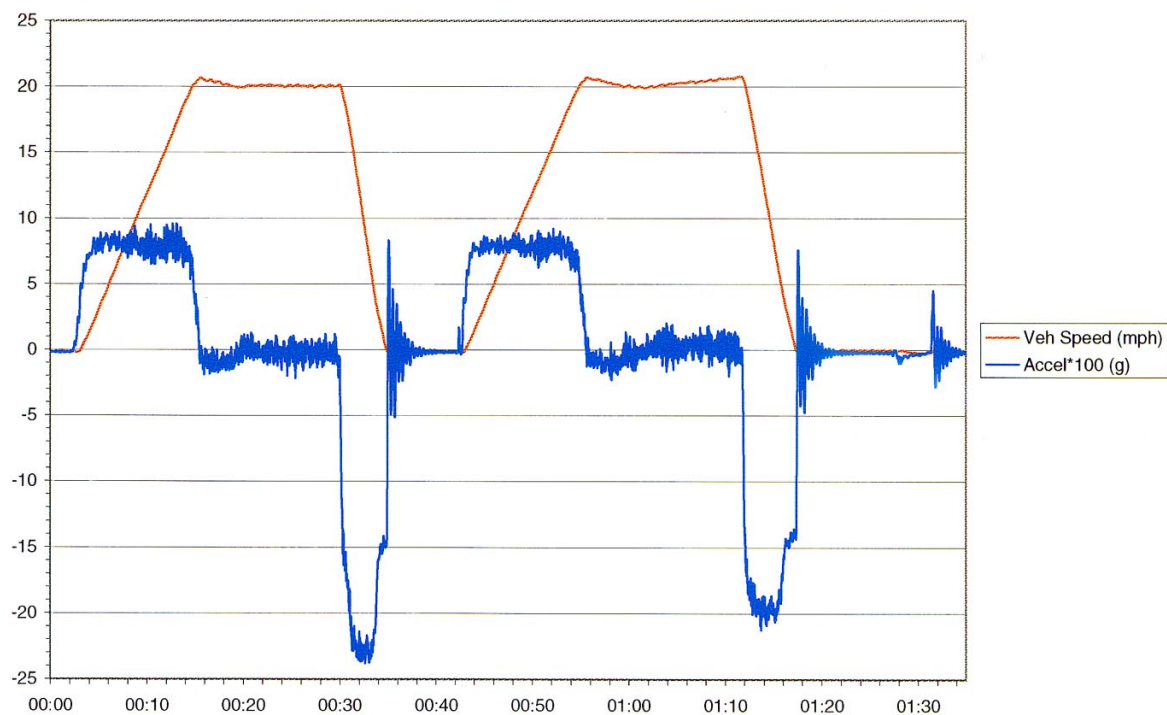




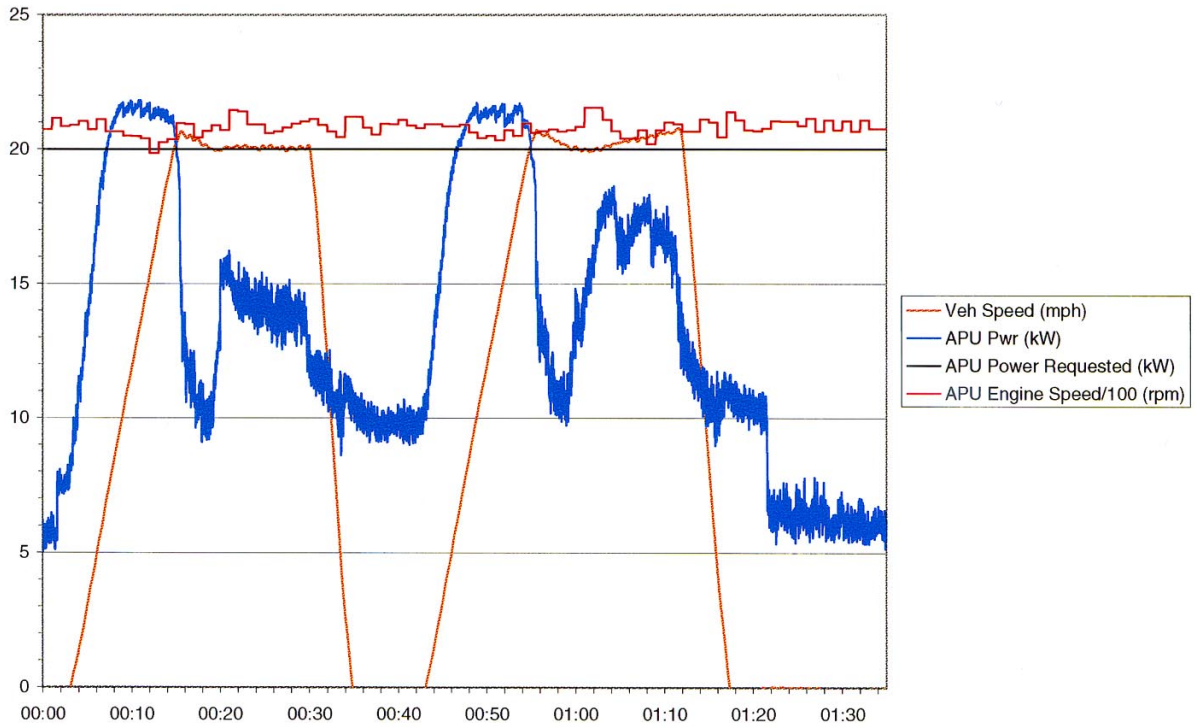
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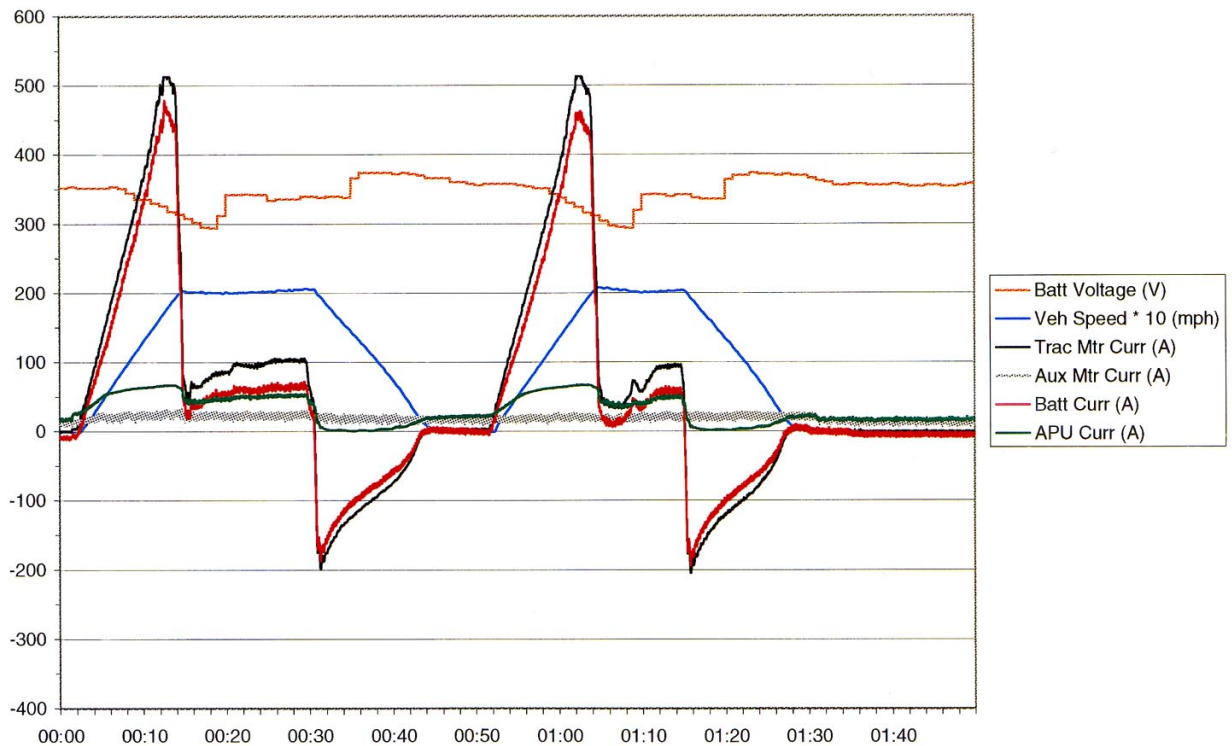
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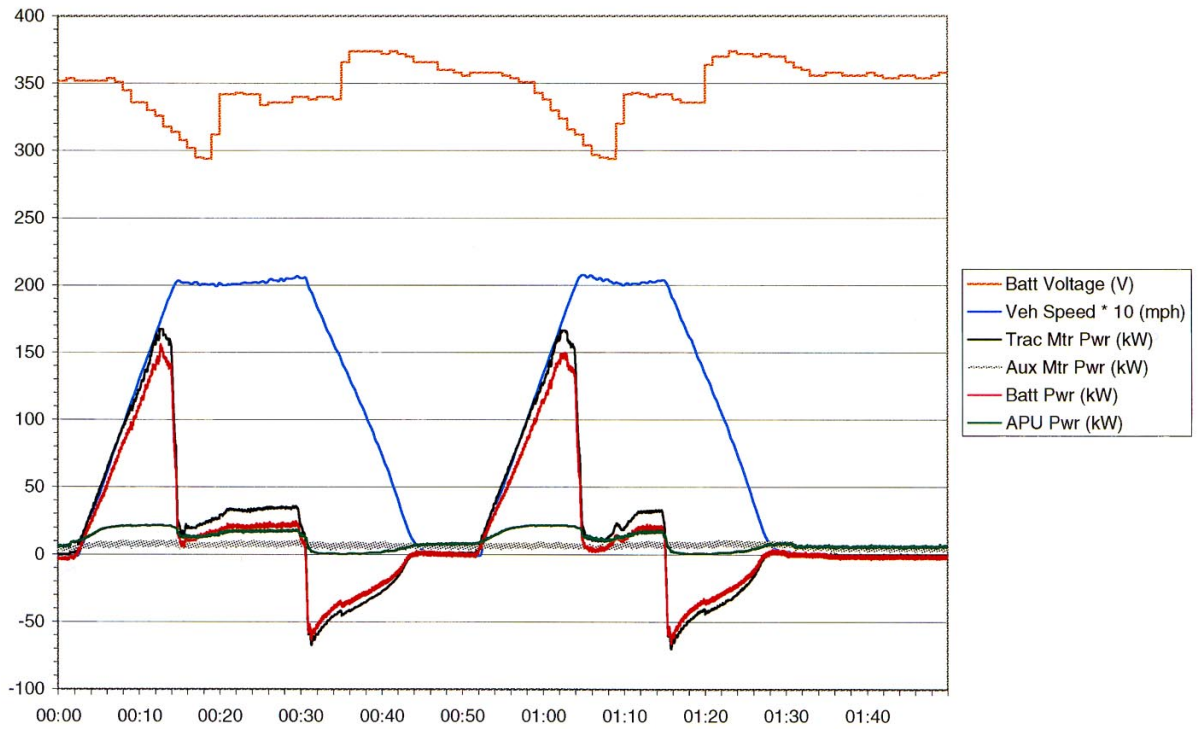
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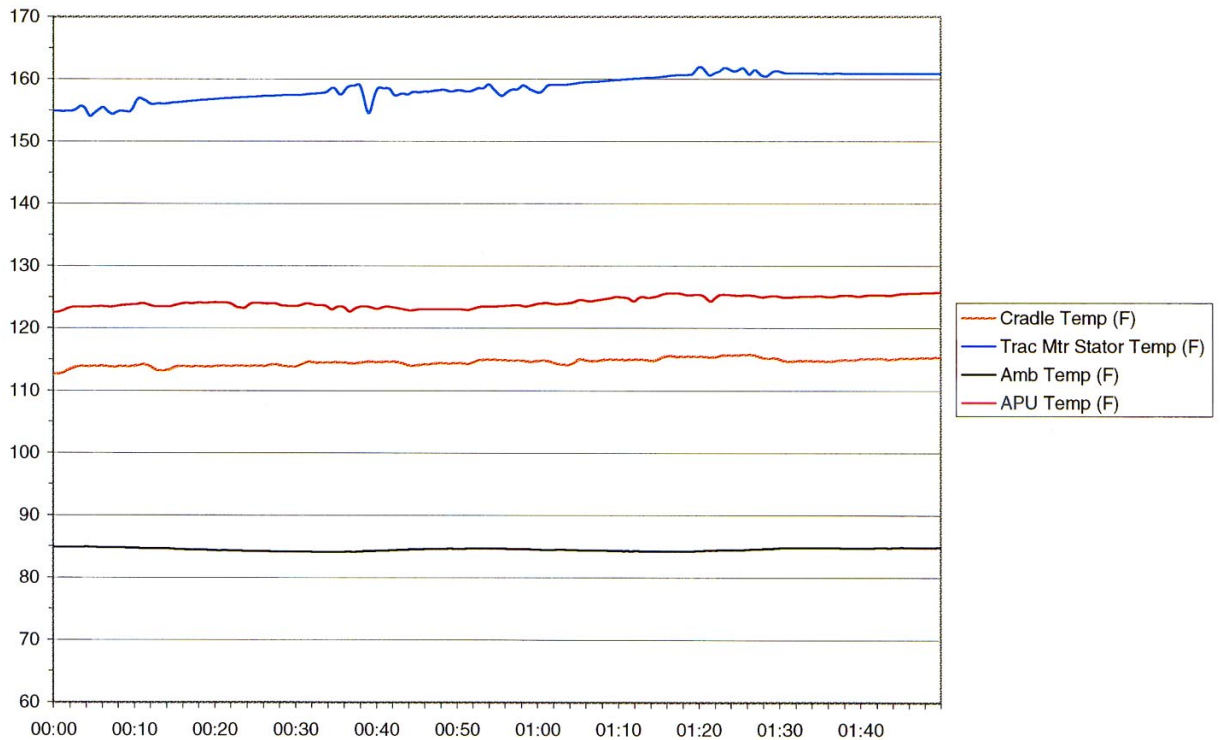
Test 8: Batteries, 0-20 mph, Regen Braking



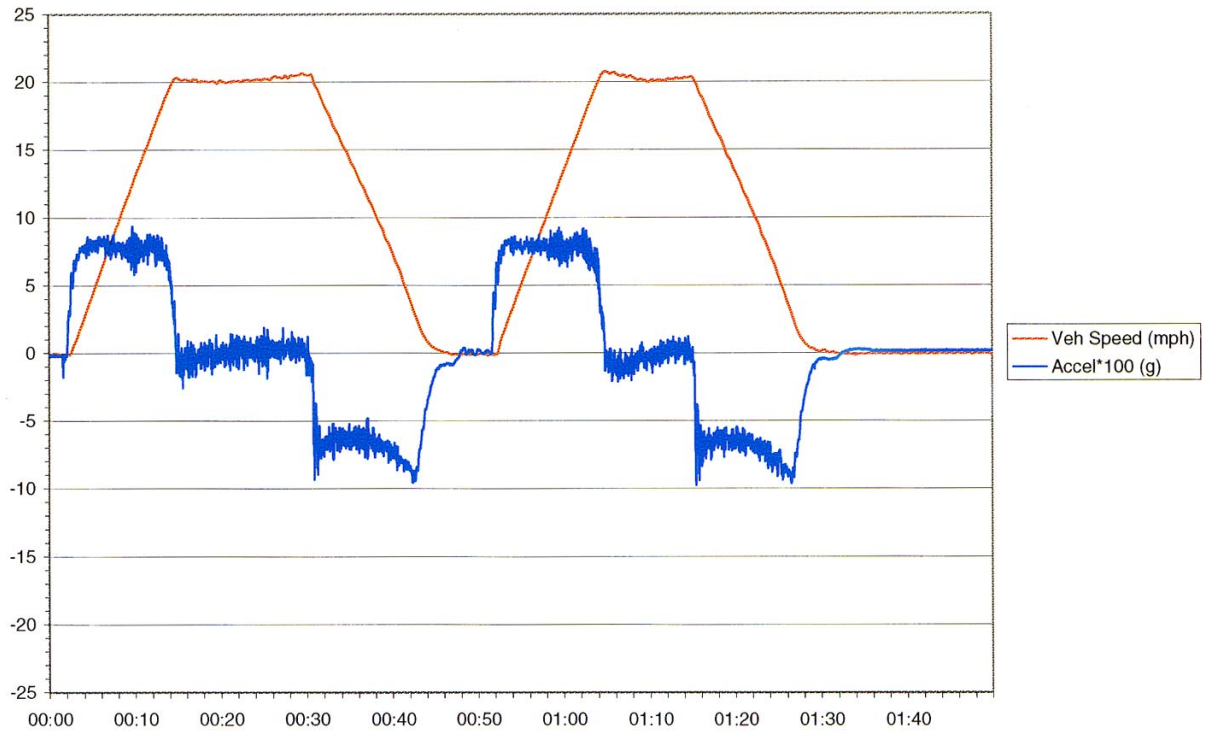
Test 8: Batteries, 0-20 mph, Regen Braking



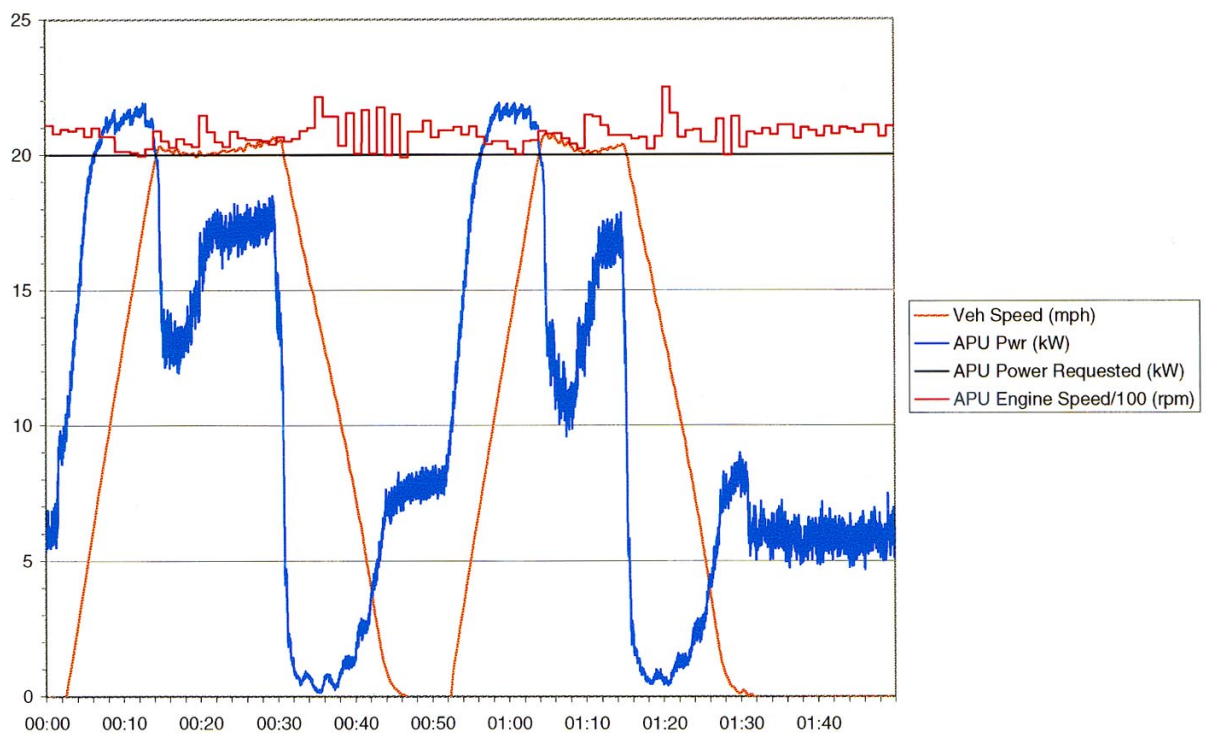
Test 8: Batteries, 0-20 mph, Regen Braking



Test 8: Batteries, 0-20 mph, Regen Braking

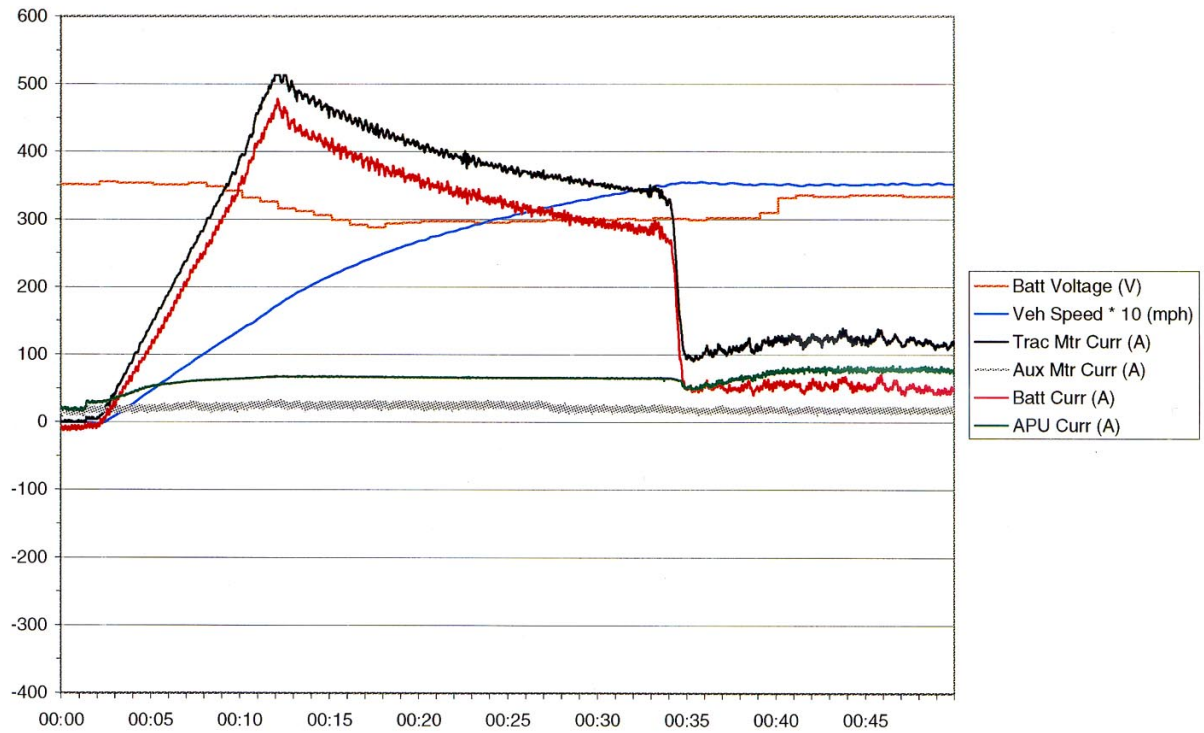


Test 8: Batteries 0-20 mph, Regen Braking

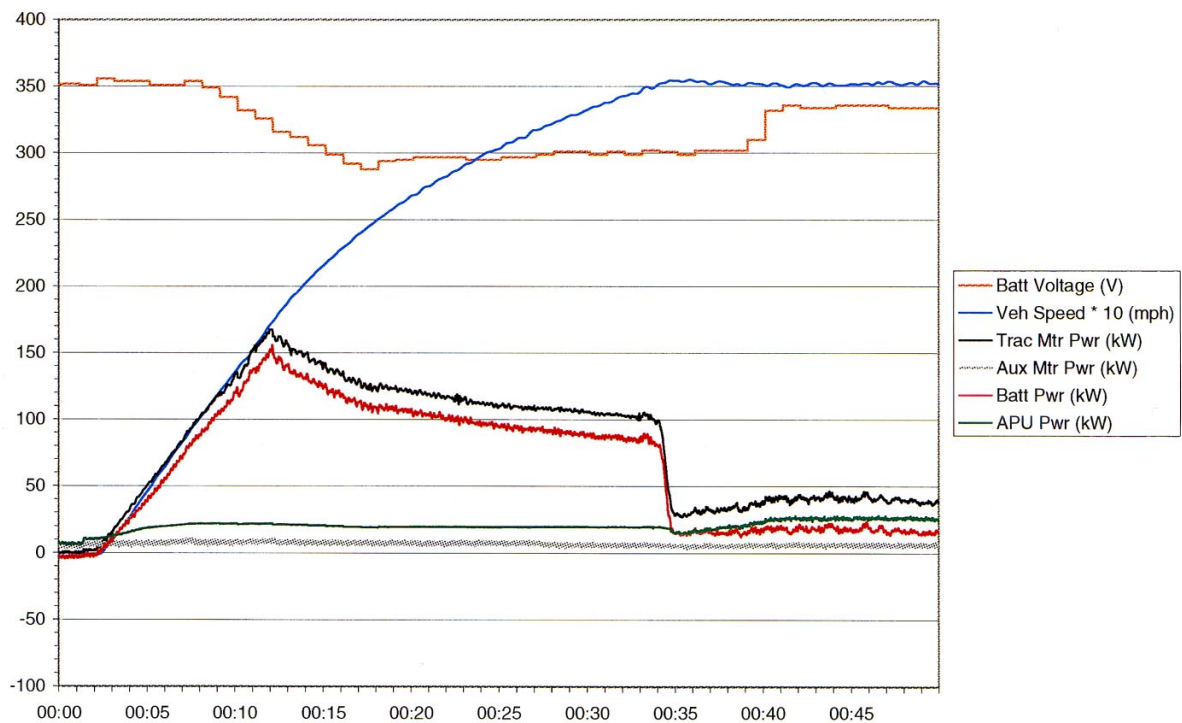




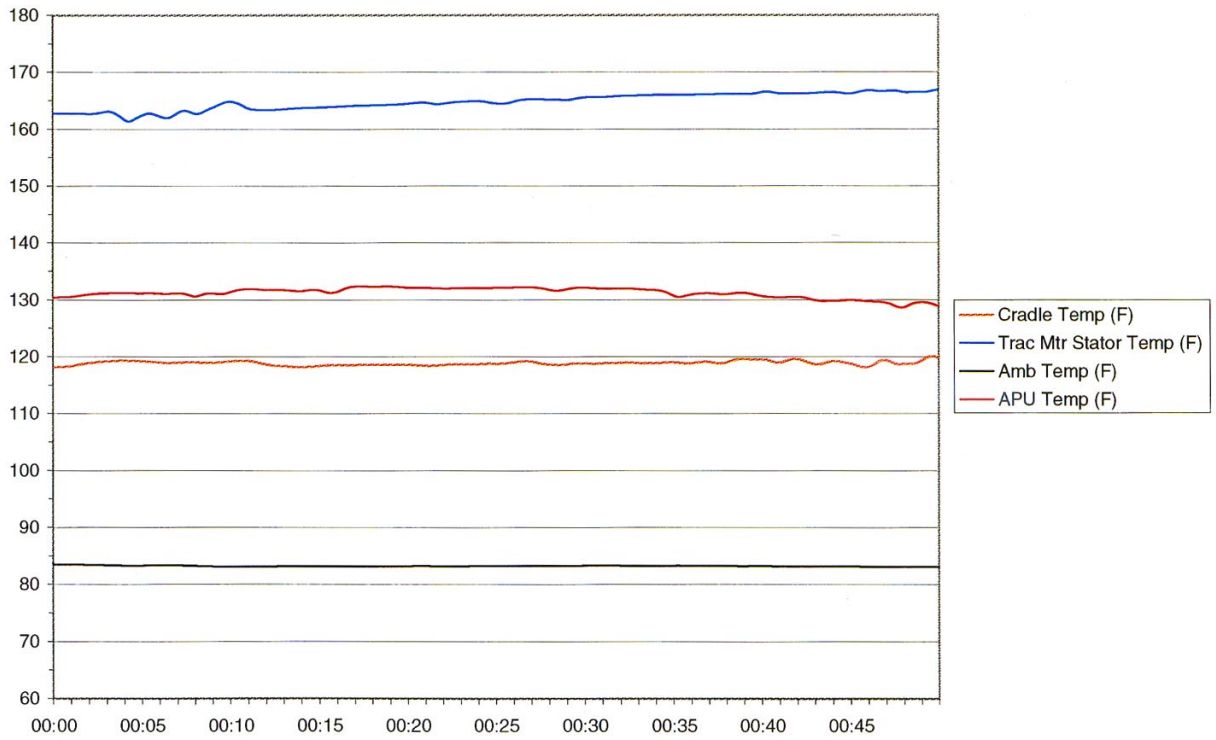
Test 9: Batteries, 0-35 mph, No Regen Braking



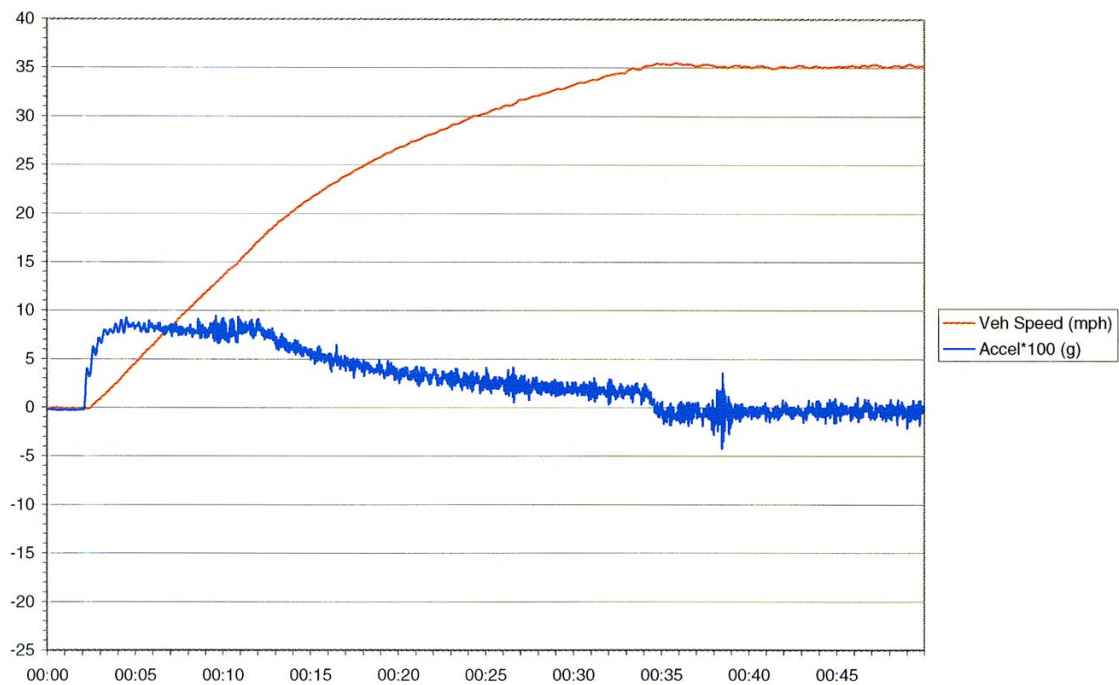
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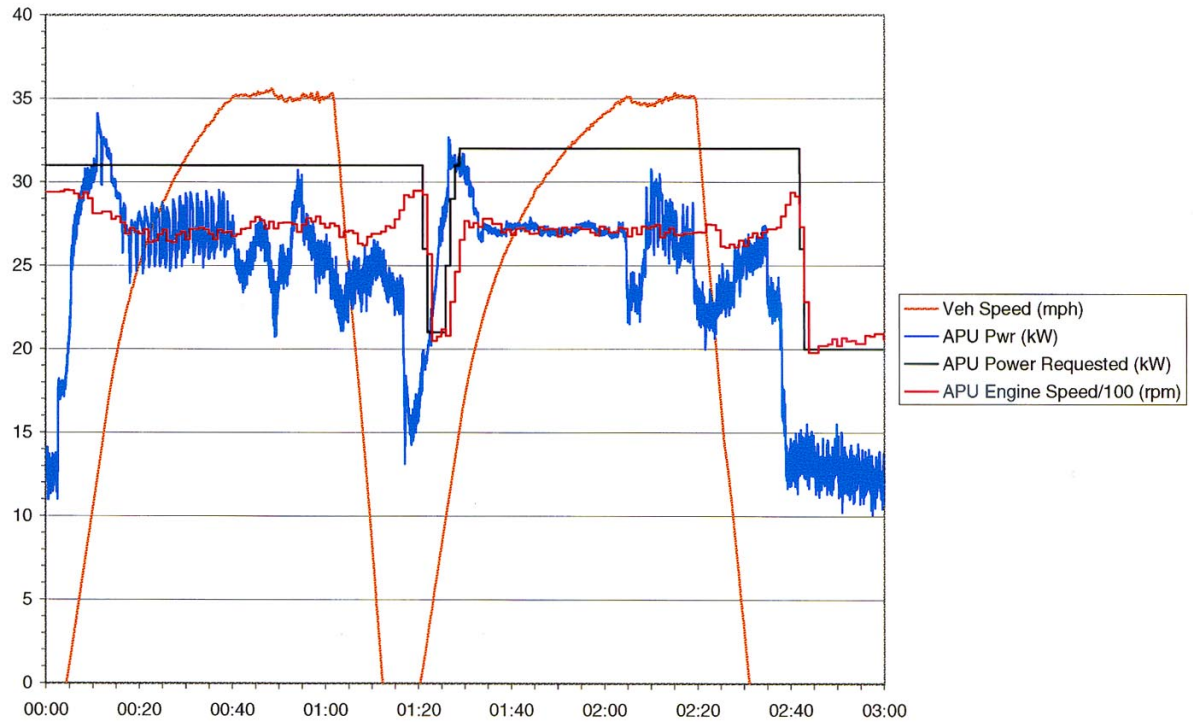
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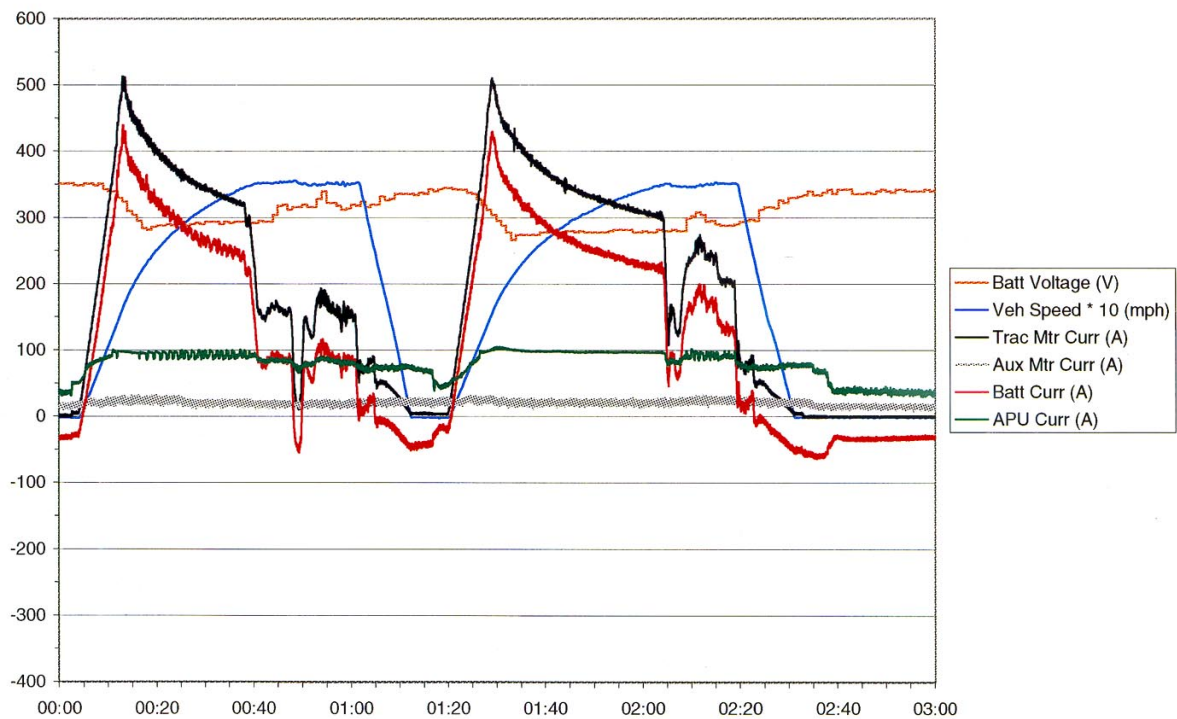
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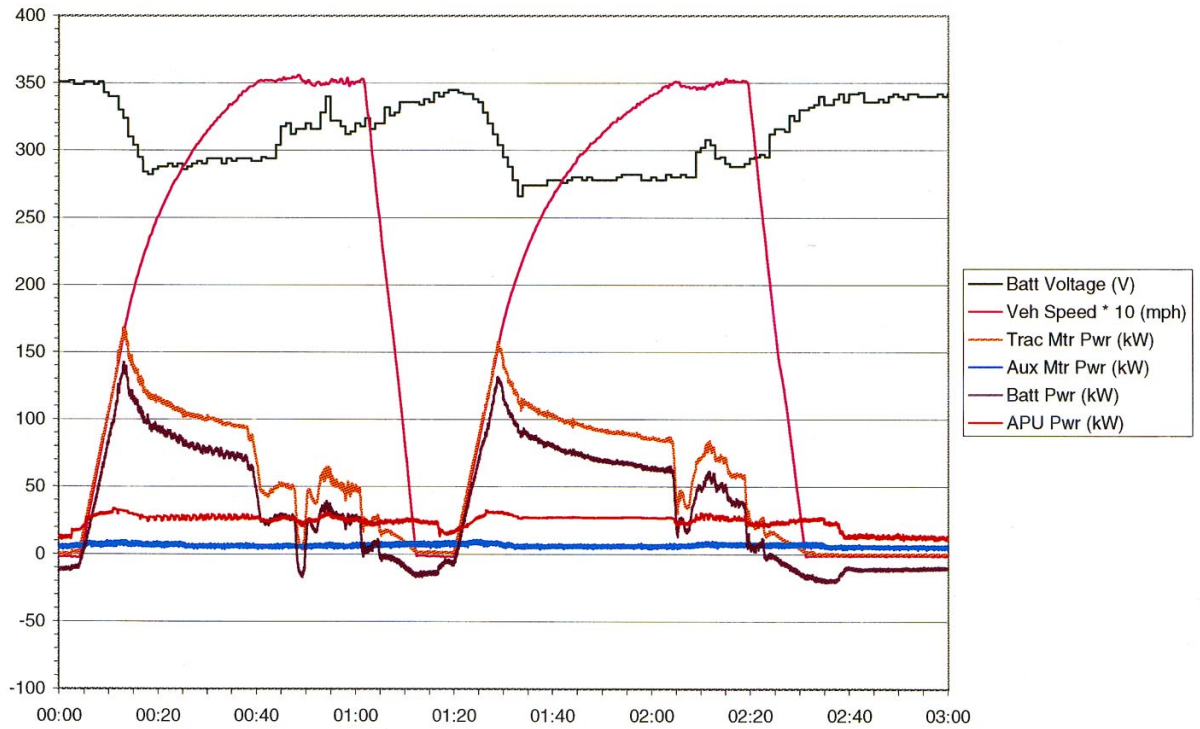
Test 9: Batteries, Arterial, 35 mph, No Regen Braking



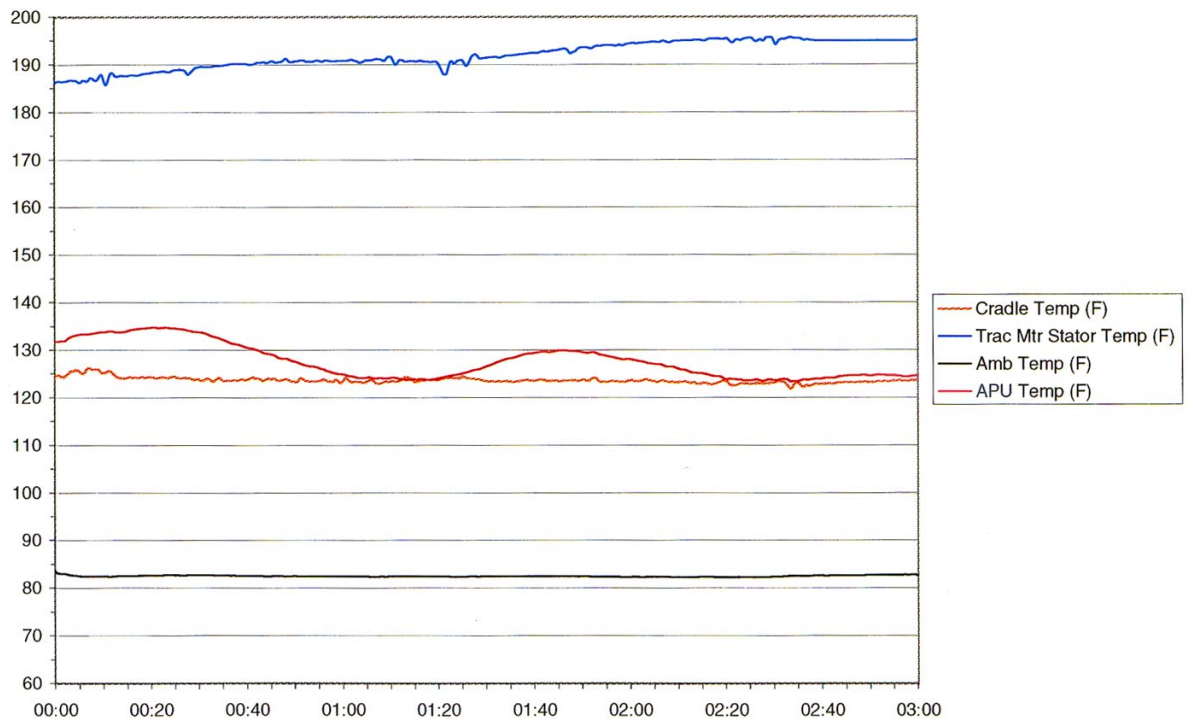
Test 10: Batteries, Arterial, 35 mph, No Regen Braking



Test 10: Batteries, Arterial, 35 mph, No Regen Braking

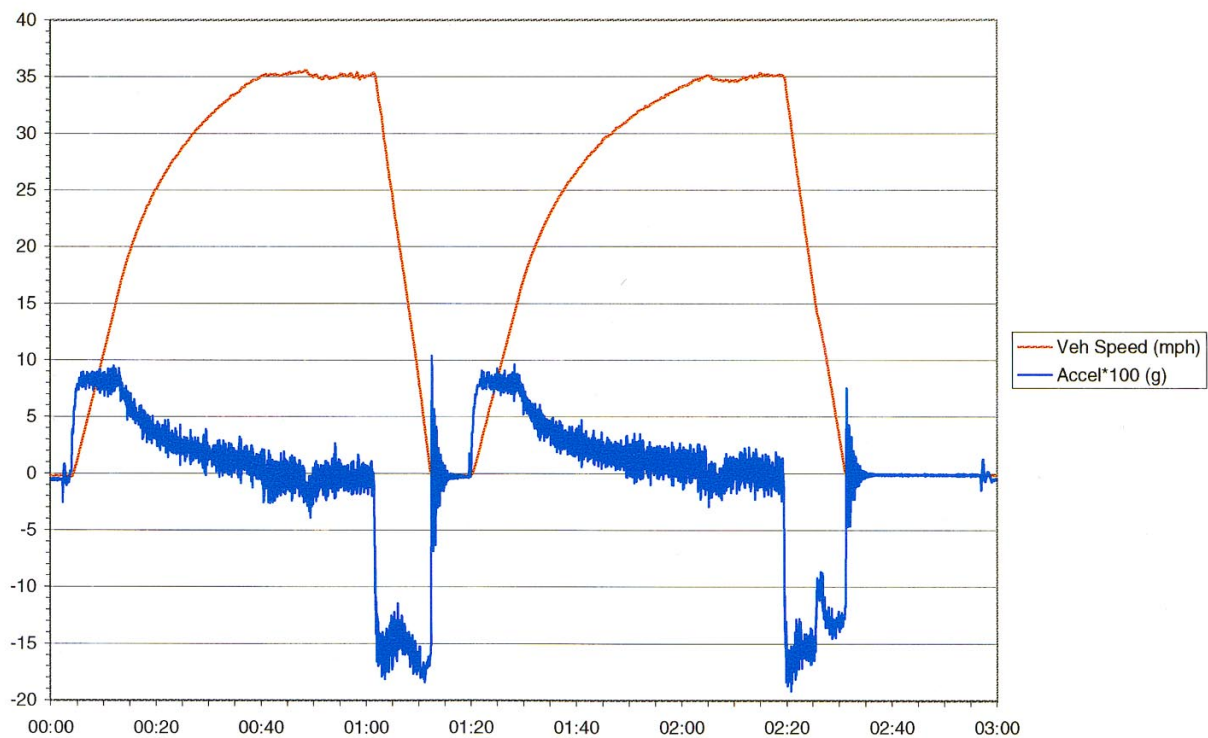


Test 10: Batteries, Arterial, 35 mph, No Regen Braking

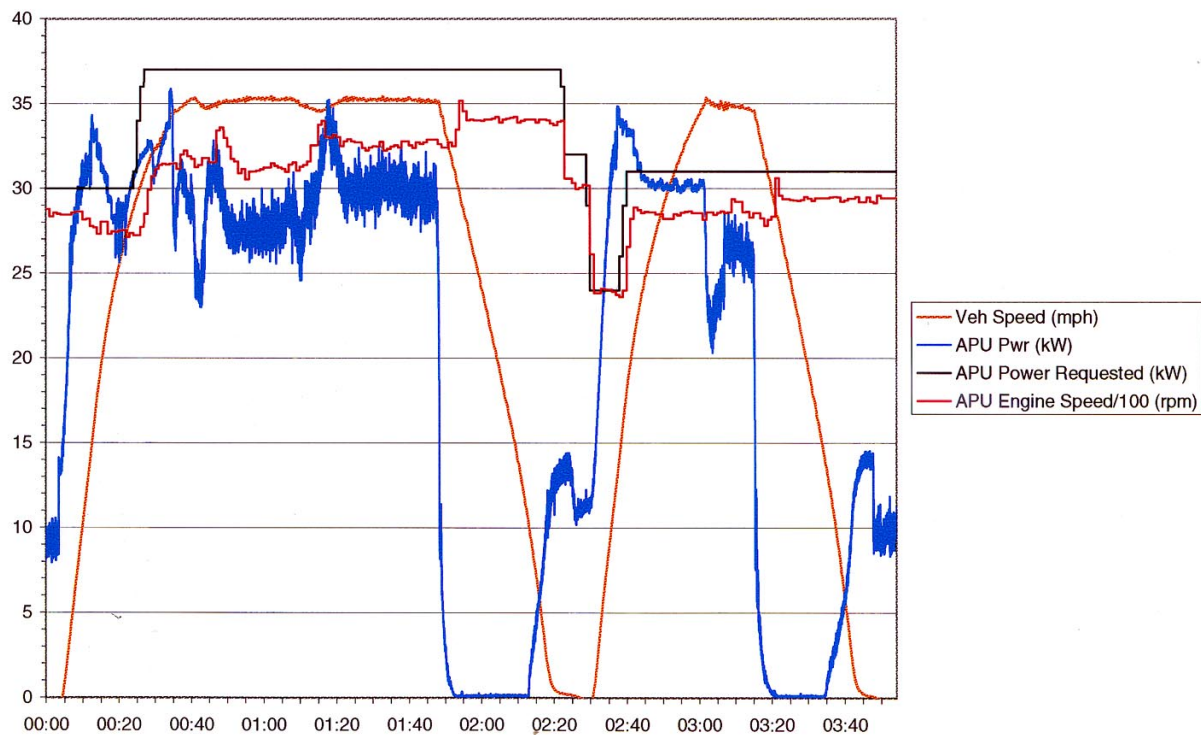




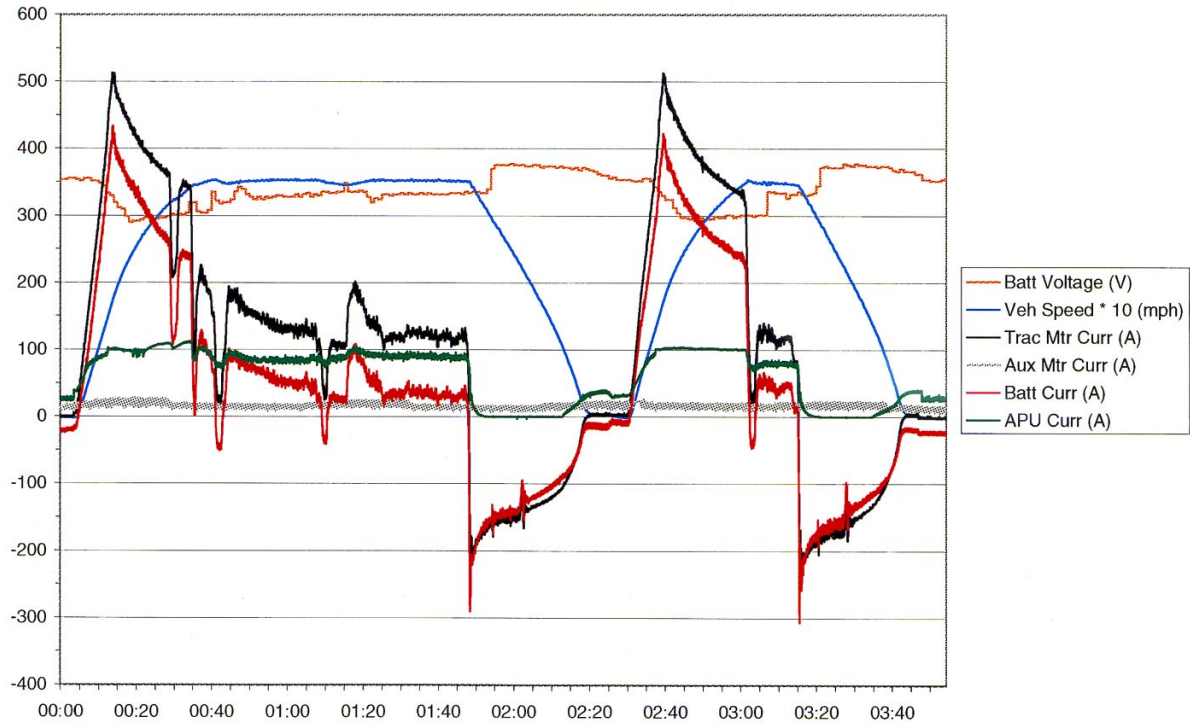
**Test 10: Batteries, Arterial, 35 mph, No Regen Braking**



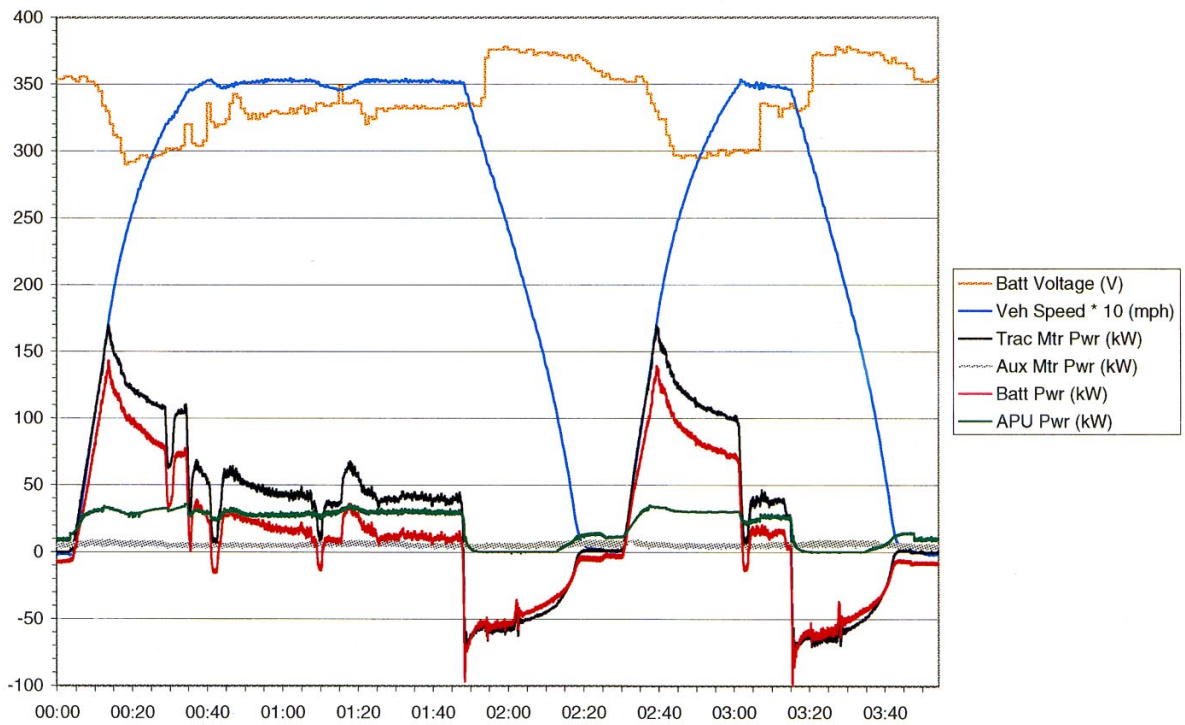
**Test 10: Batteries, Arterial, 35 mph, Regen Braking**



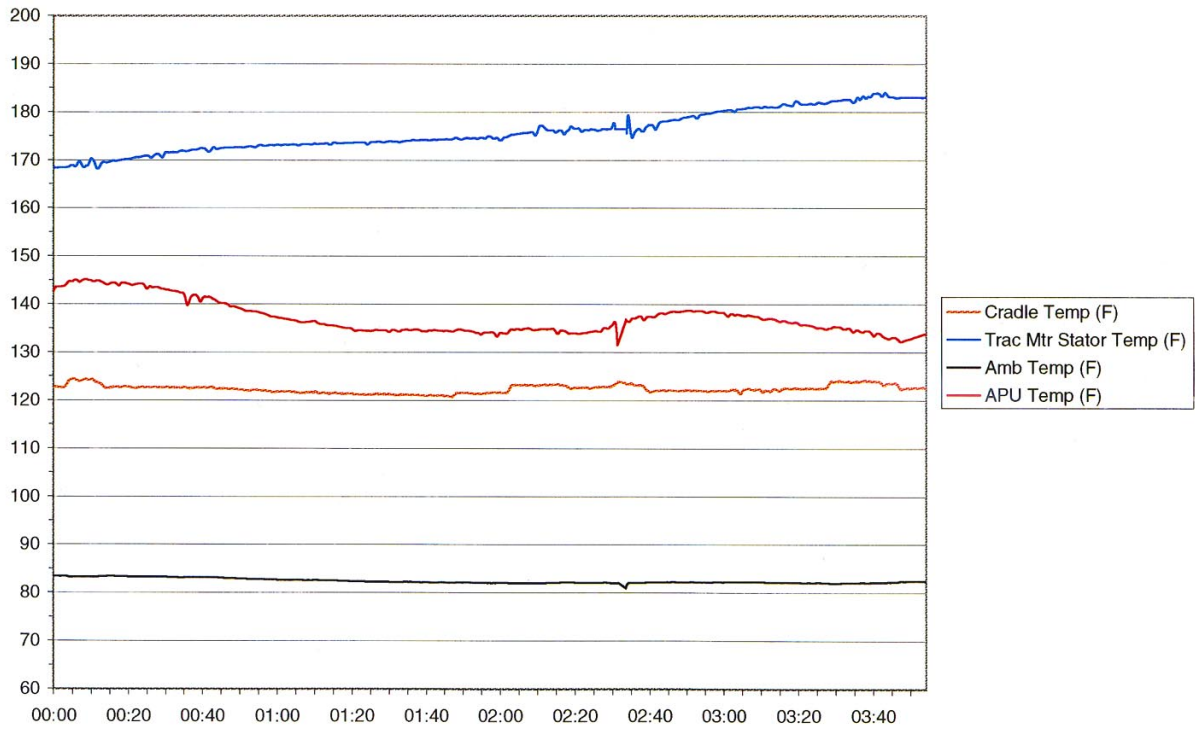
Test 11: Batteries, Arterial, 35 mph, Regen Braking



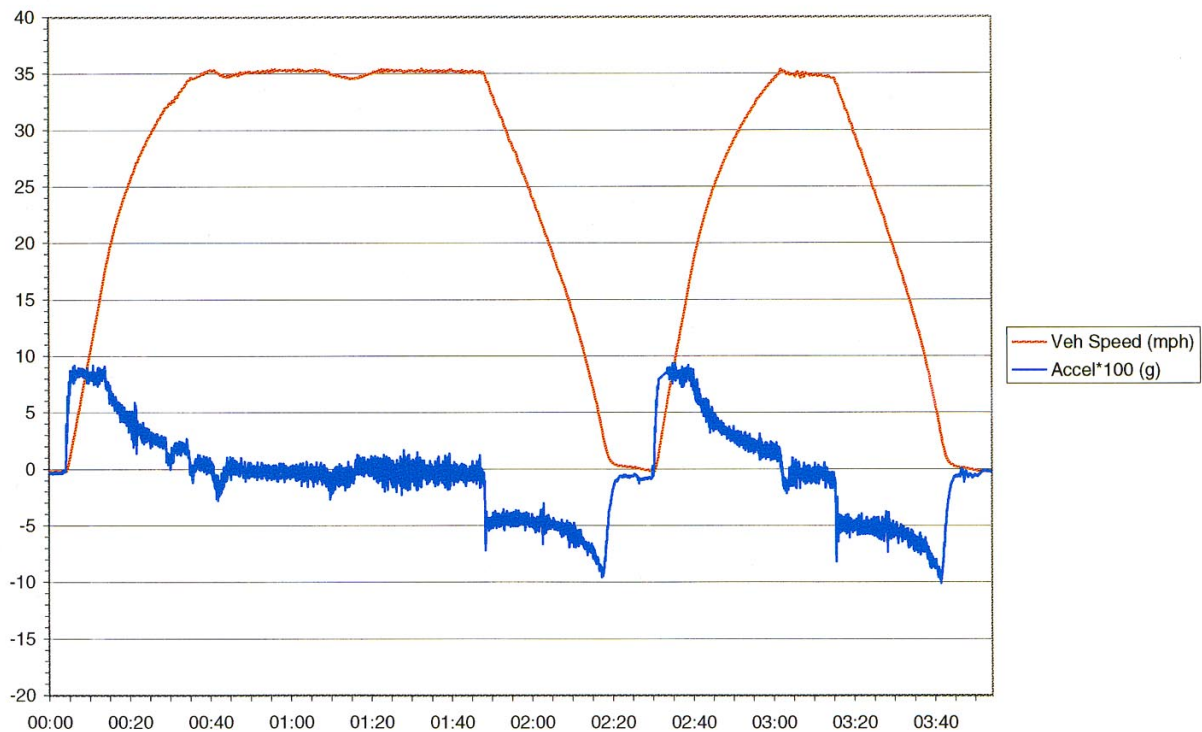
Test 11: Batteries, Arterial, 35 mph, Regen Braking



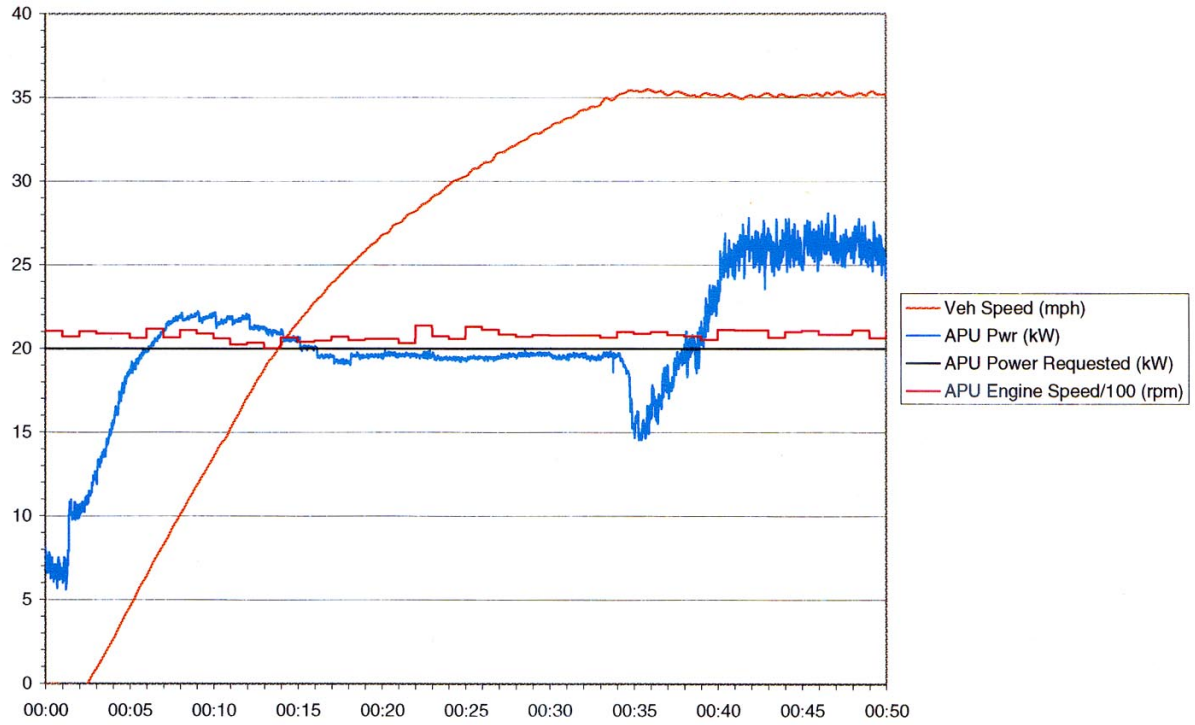
Test 11: Batteries, Arterial, 35 mph, Regen Braking



Test 11: Batteries, Arterial, 35 mph, Regen Braking



Test 11: Batteries, 0-35 mph, No Regen Braking





## APPENDIX F

### DESCRIPTION OF CONTROL ALGORITHM FOR THE CAPACITOR ENERGY STORAGE SYSTEM

The capacitor energy storage system is controlled using proportional-integral (PI) control. The capacitor voltage is the control variable. It is acquired once per second from the APU by the laptop PC. The output variable is the APU power requested. The laptop and APU controller communicate via a controller area network (CAN) interface.

#### Basic Control Algorithm

At time  $t=k$ ,

1. Acquire capacitor voltage,  $V_{cap}(k)$ , from the APU.
2. Calculate the normalized error at time  $k$ ,  $e(k)$ , by comparing the capacitor voltage to the desired setpoint voltage,  $V_{sp}$ . That is,  $e(k) = (V_{sp} - V_{cap}) / V_{max}$ , where  $V_{max}$  is the maximum full scale system voltage, taken to be 400 V. The normalized error is also saved as  $e(k-1)$  for the next calculation. The setpoint used for testing was 325 V.
3. Compute the output variable at time  $k$ ,  $P(k)$ , with PI compensation, with the equation  $P(k) = bP(k-1) + K_c [(1+b)/(1+a)] [e(k) - ae(k-1)]$ , where  $K_c$  is the controller gain,  $a$  is the reset time coefficient ( $0 < a < 1$ ) and  $b$  is the integration constant ( $b = 0$  or  $1$ ). The values used during the testing were  $K_c = 1.5$  kW/V,  $a = 0.95123$  and  $b = 1$ . These parameters were determined from the response characteristics of the APU and the sampling rate of 1 Hz. The output variable is saved for use as  $P(k-1)$  in the next calculation.
4. Limit  $P(k)$  between 0 and the maximum output APU power, which is 39 kW, and submit the power request to the APU via CAN.
5. Repeat steps 1-4 for time  $T = k+1, k+2$ , etc.

#### Enhancements to the Basic Control Algorithm

Automatic control mode uses the algorithm described above. Manual control mode allows the user to input an APU power setpoint directly. In order to provide “bumpless” transfer between modes, the following steps are taken.

When transitioning from manual to automatic mode, the past value of the output variable,  $P(k-1)$ , is set to the current value of the manual mode power setpoint. The past error value,  $e(k-1)$ , is set to 0. The controller setpoint value,  $V_{sp}$ , is set to the current value of the capacitor voltage.

When transitioning from automatic to manual mode, the last value of the output variable,  $P(k)$ , is used as the initial value of the manual mode APU power setpoint.

Two additions to the basic control algorithm were made in order to optimize the operation of the APU. Engines run most efficiently when run at constant speed, especially near their “sweet spot”. Quantization and peak holding are two ways of

minimizing the number of changes in APU power requested, and, therefore, the changes in rpm.

Quantization forces the output to assume one of a discrete set of equally spaced output values rather than providing for a continuum of possible output values. The number of quantization levels,  $n$ , becomes another system parameter. The effect on the system is to eliminate minor fluctuations in engine speed. A value of  $n = 20$  was used during the capacitor tests described in this report.

Peak holding introduces a delay between the determination of the need for a decrease in APU power and its actual implementation by a time,  $T$ . Increases in power are not delayed and they supersede any pending decreases, i.e., increases in power reset the counter used to implement the delay for decreases in power. This provides more nearly constant APU operation during busy start/stop cycles without affecting acceleration performance. Peak holding was not used during the tests described in this report. A typical value for  $T$  would be 5 sec.

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13. ABSTRACT (Maximum 200 words)  A government, industry and academic cooperative has developed a Hybrid Electric Transit Bus (HETB). Goals of the program include doubling the fuel economy of city transit buses currently in service, and reducing emissions to one-tenth of EPA standards. Unique aspects of the vehicle's power system include the use of ultra-capacitors for the energy storage system and the planned use of a natural gas fueled turbogenerator, to be developed from a small jet engine. At over 17000 kg gross weight, this is the largest vehicle to use ultra-capacitor energy storage. A description of the HETB, the results of performance testing, and future vehicle development plans are the subject of this report.				
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